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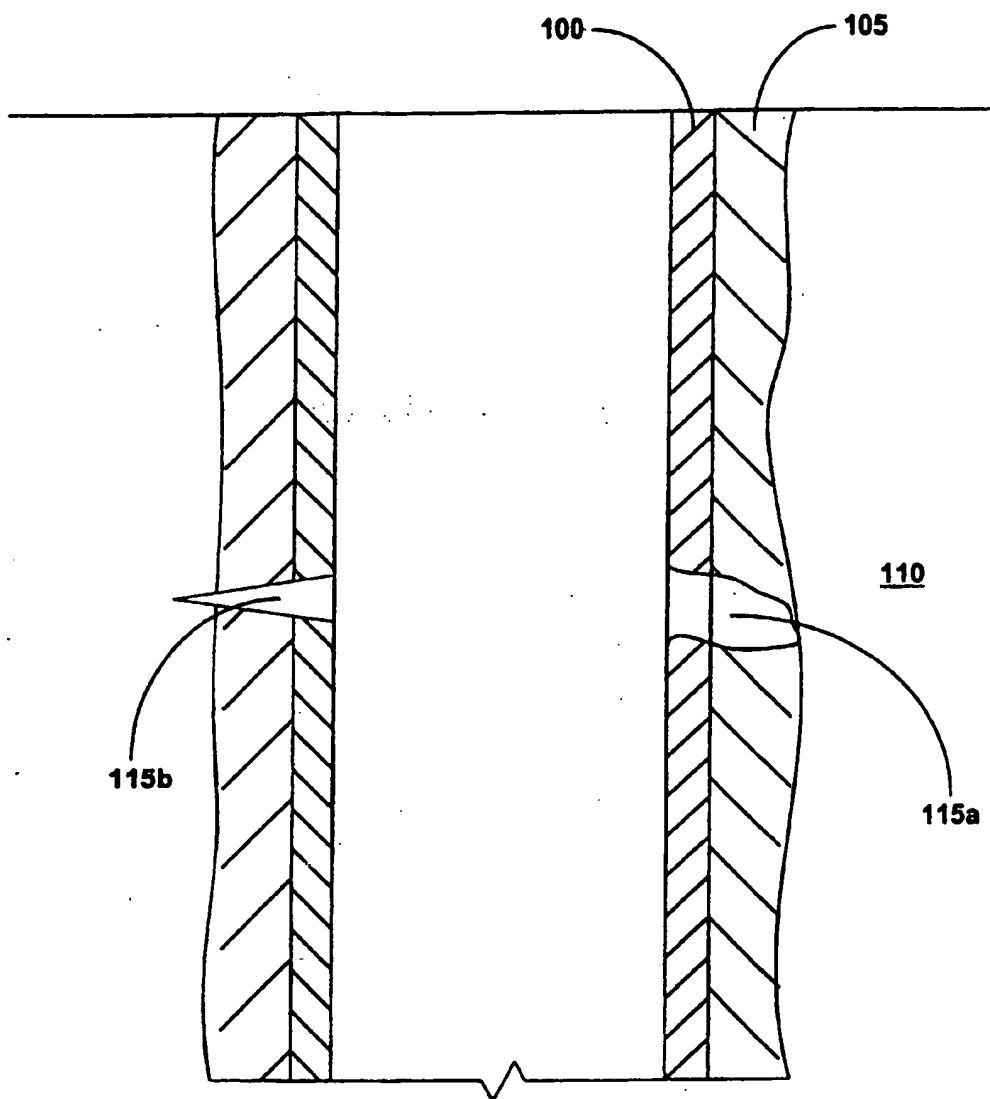


FIGURE 1

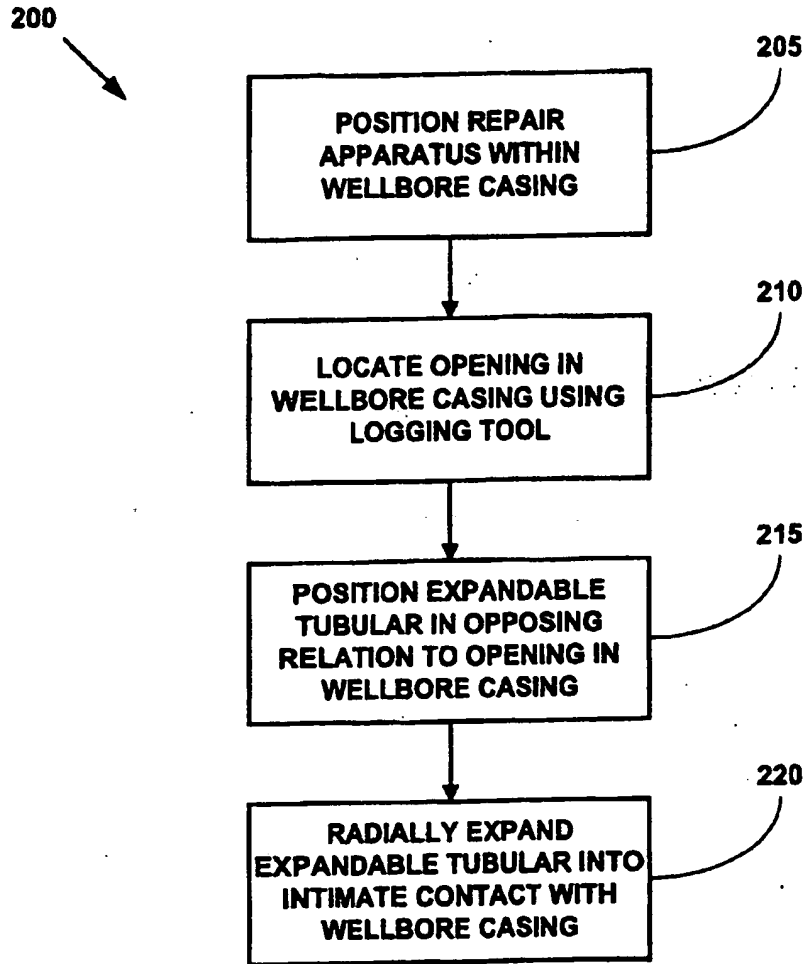


FIGURE 2

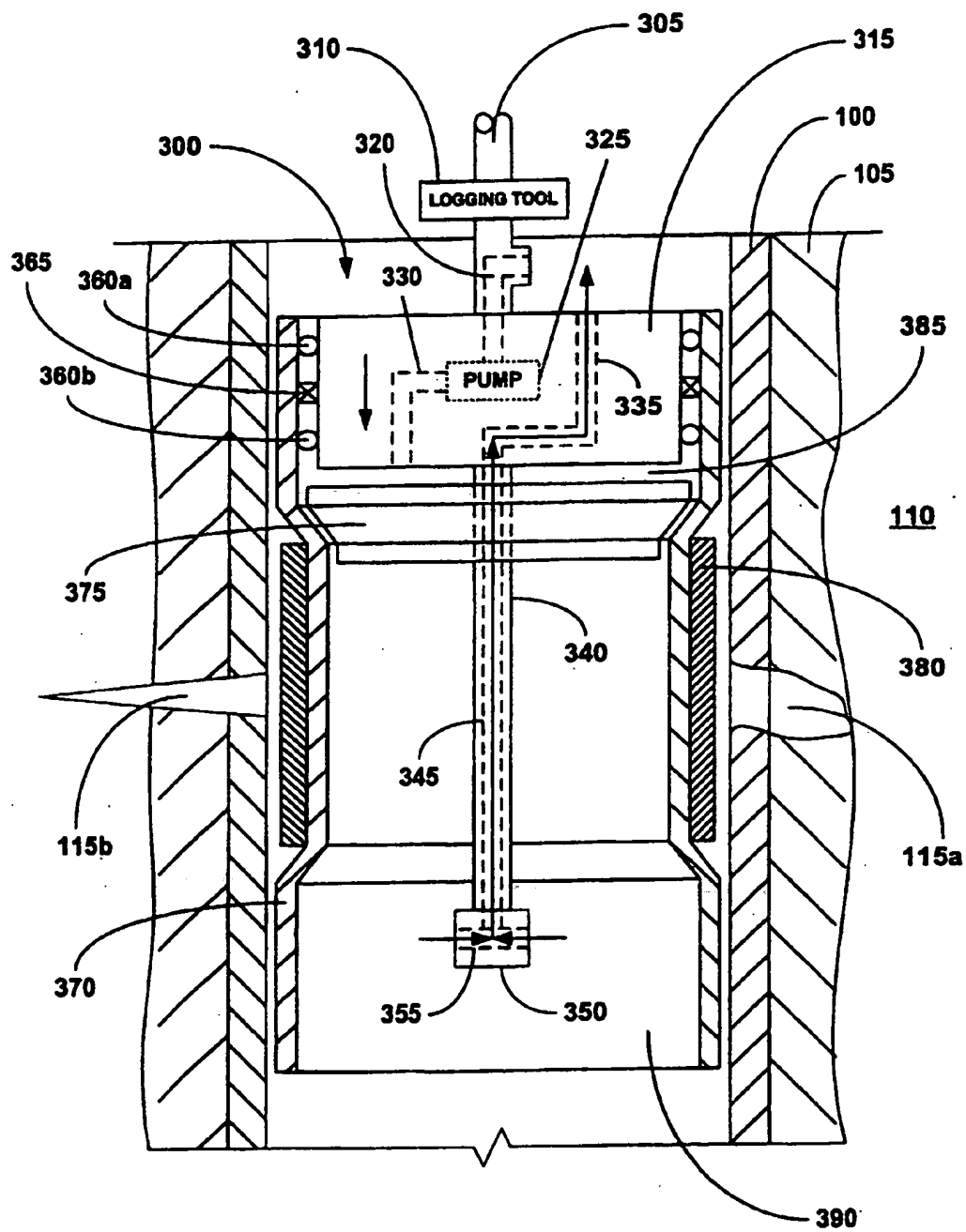


FIGURE 3a

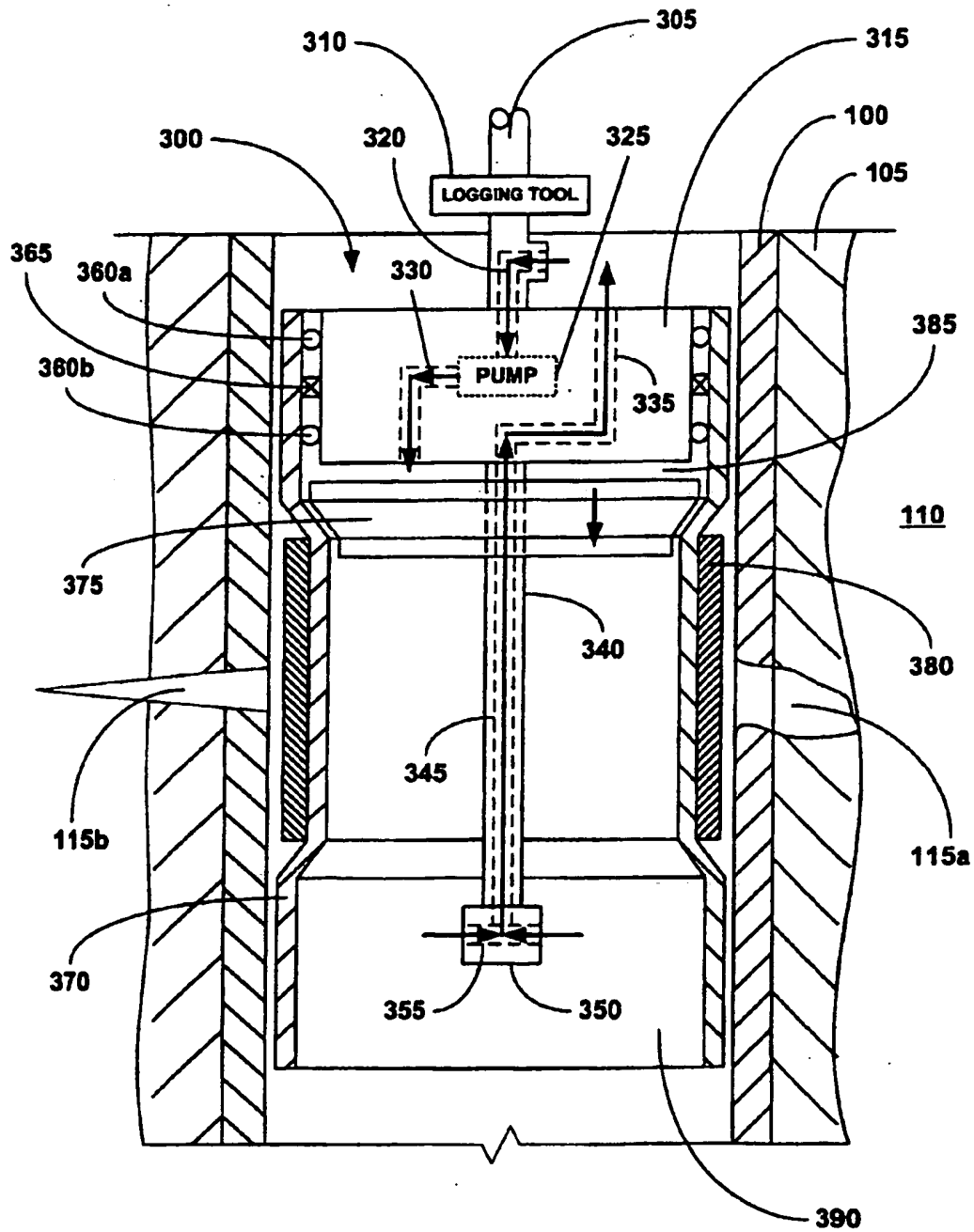


FIGURE 3b

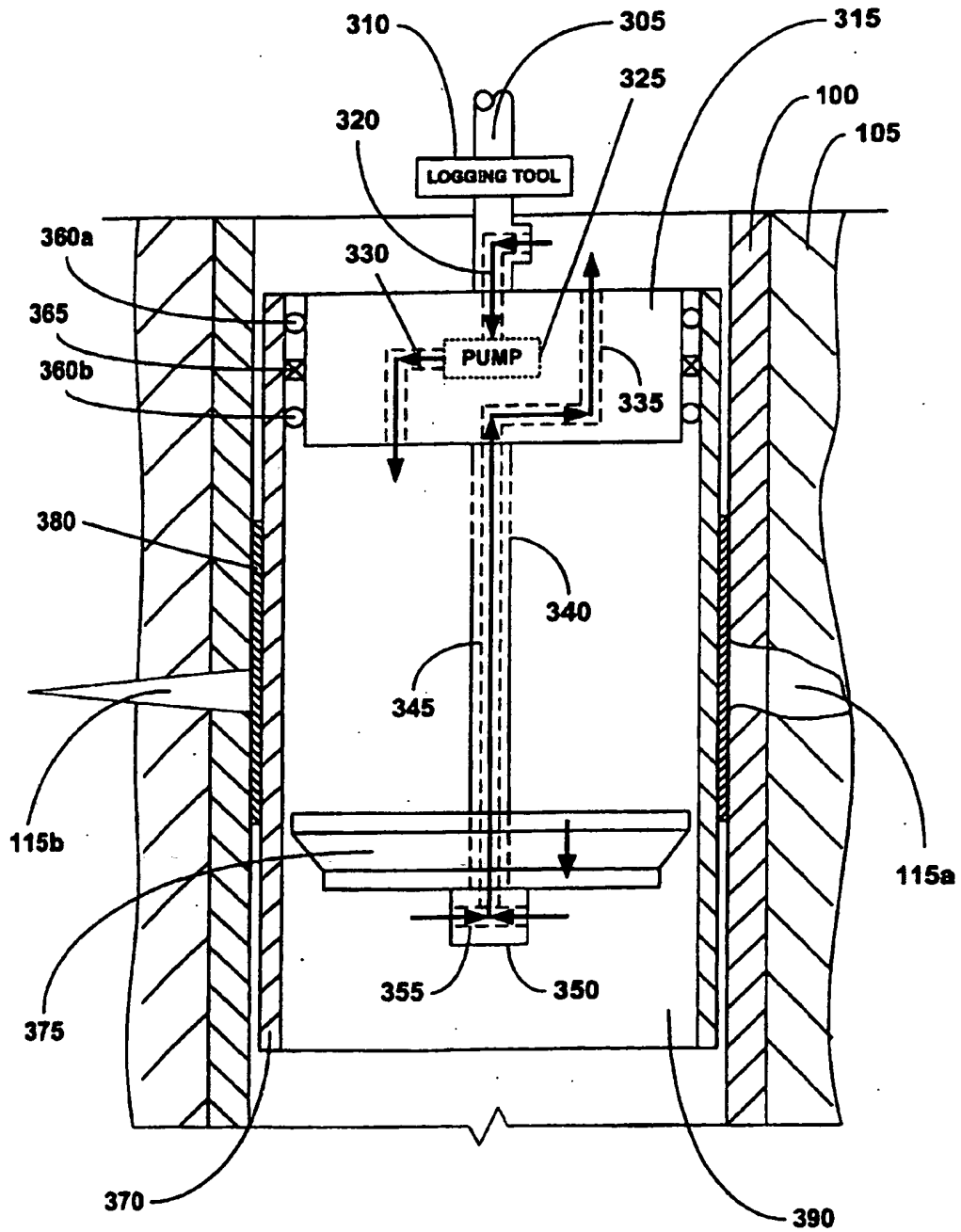


FIGURE 3c

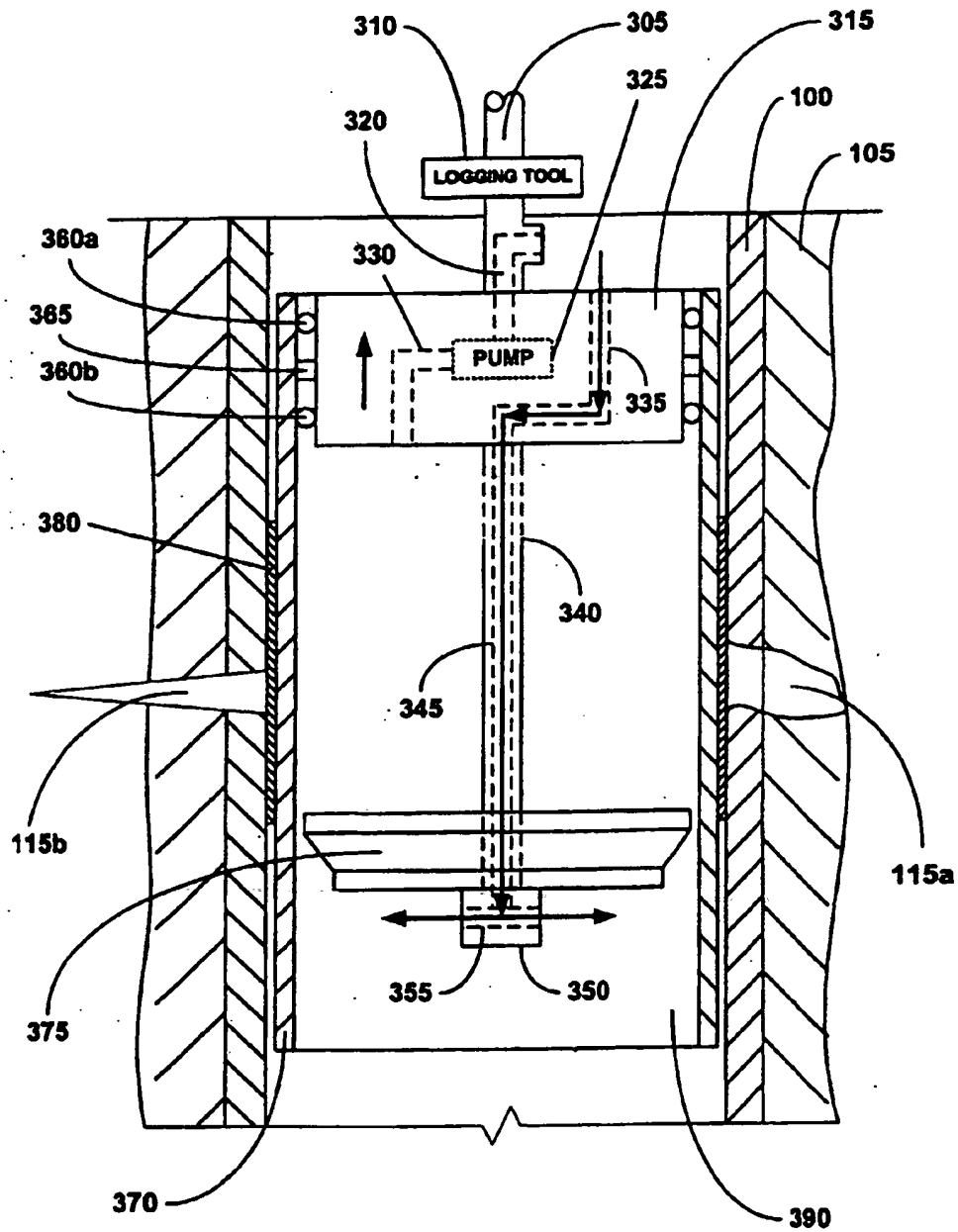


FIGURE 3d

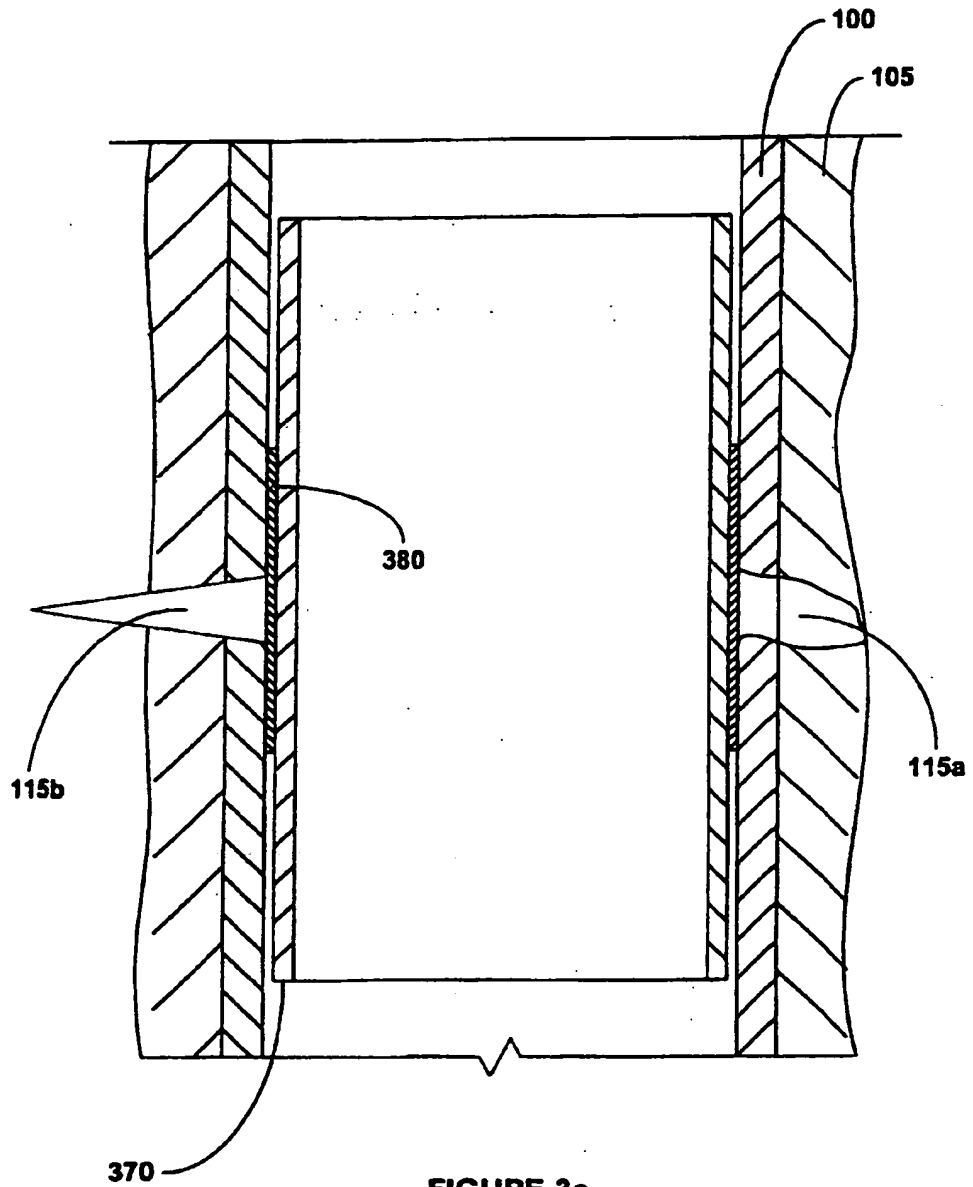


FIGURE 3e



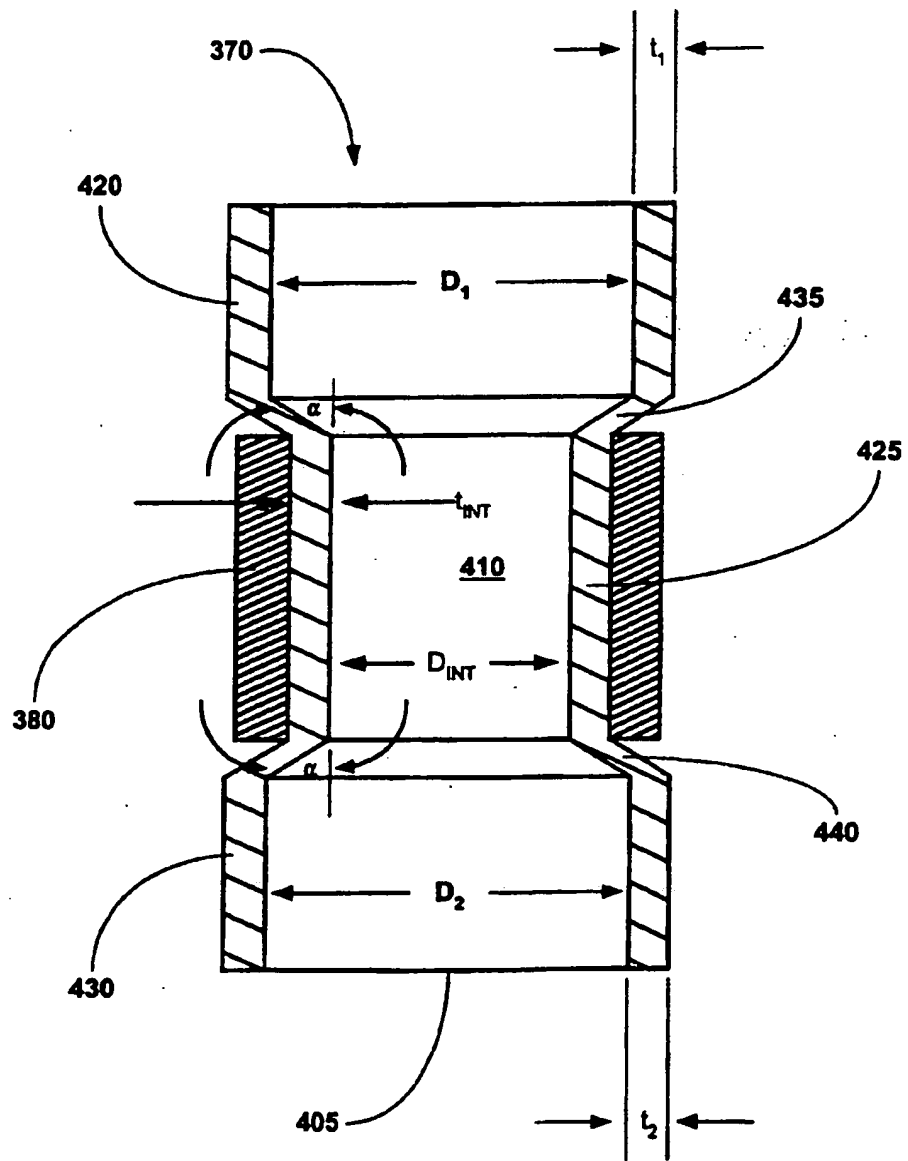


FIGURE 4

500

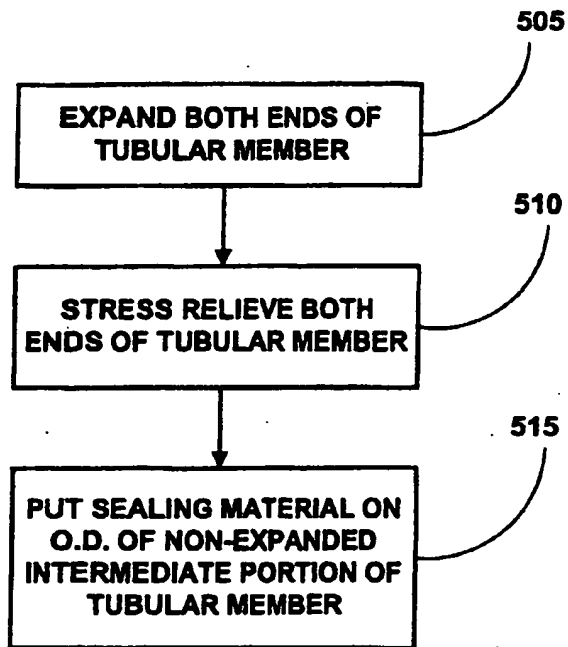


FIGURE 5

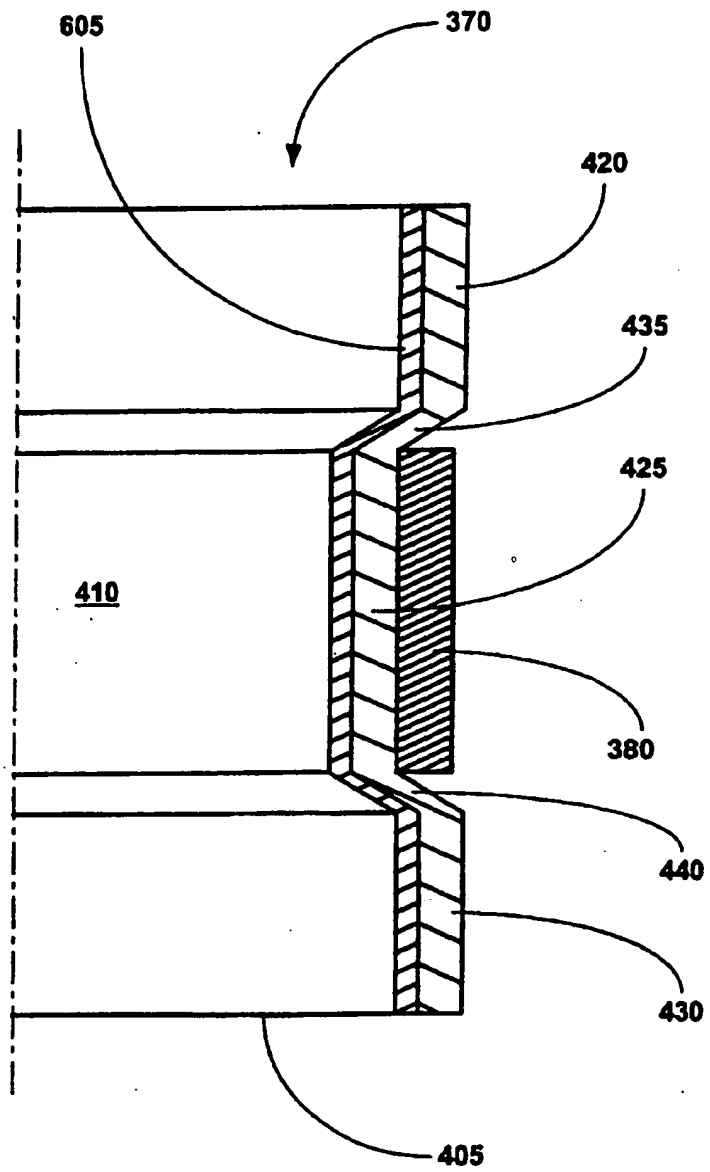


FIGURE 6

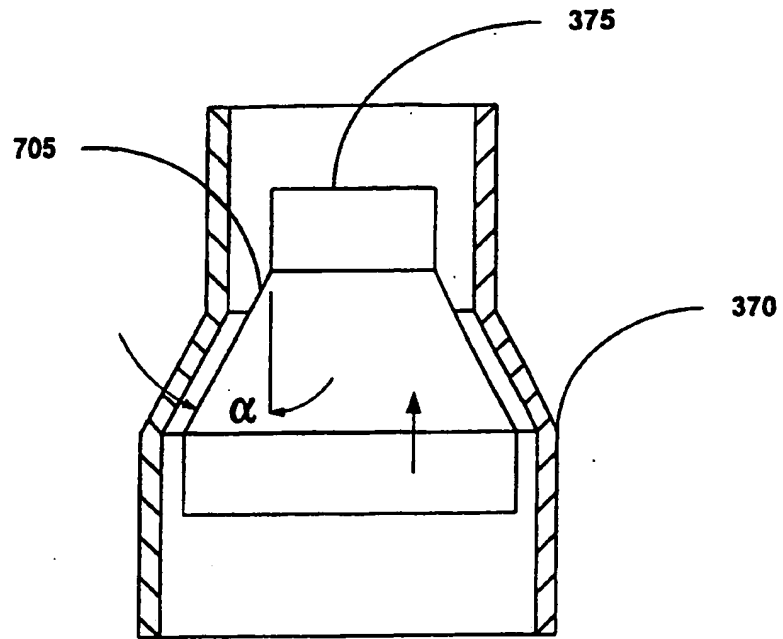


FIGURE 7

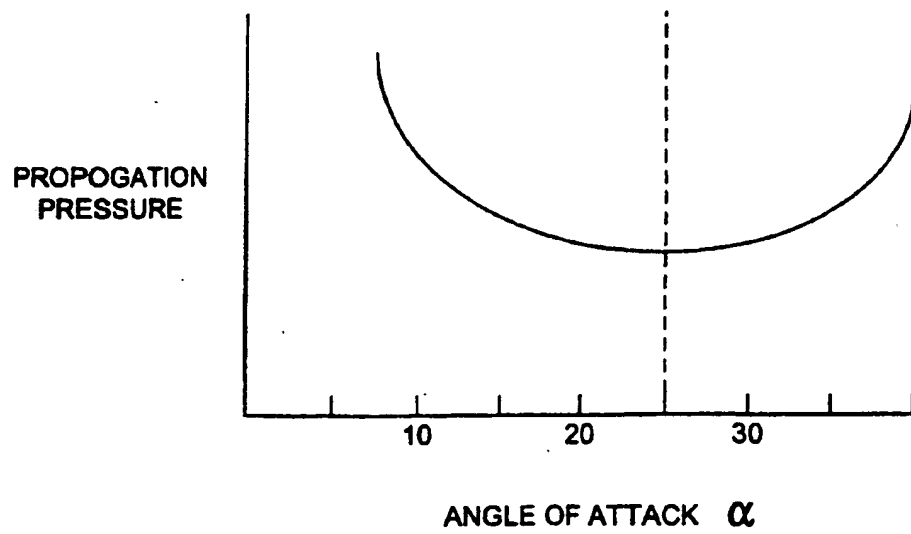


FIGURE 8

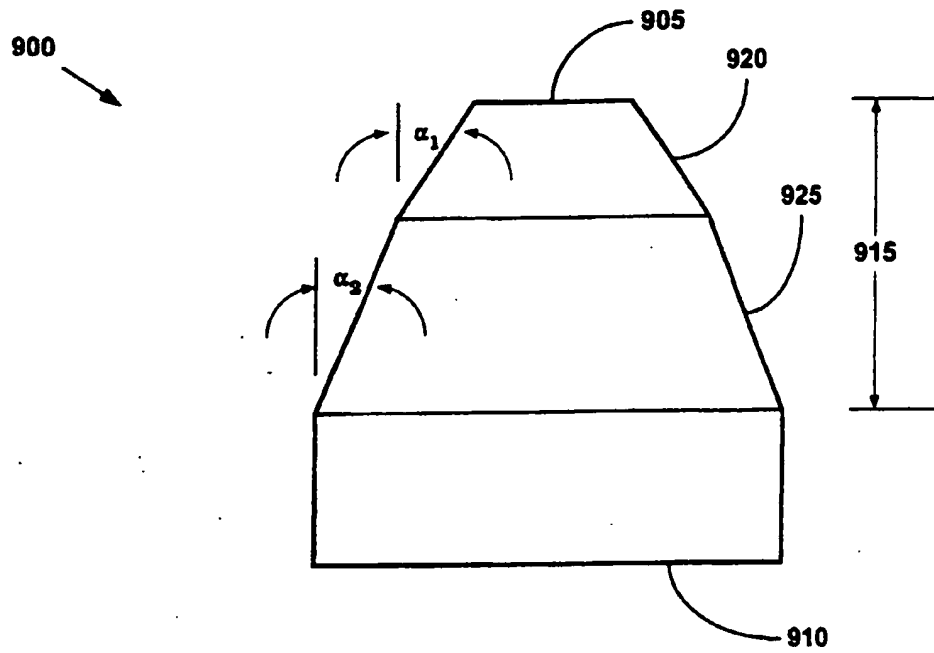
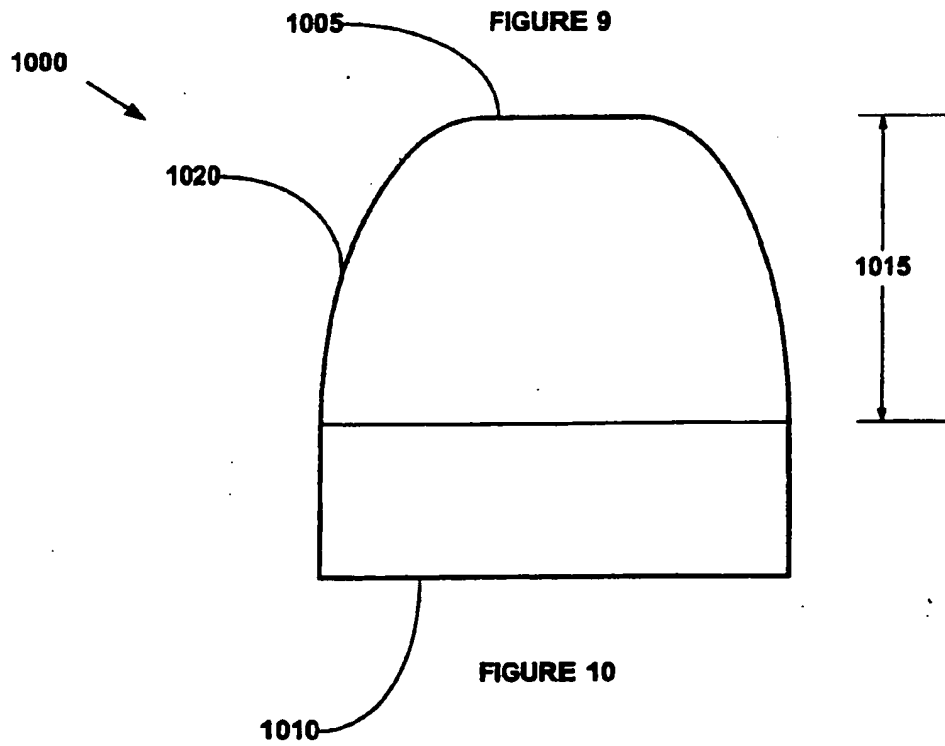


FIGURE 9



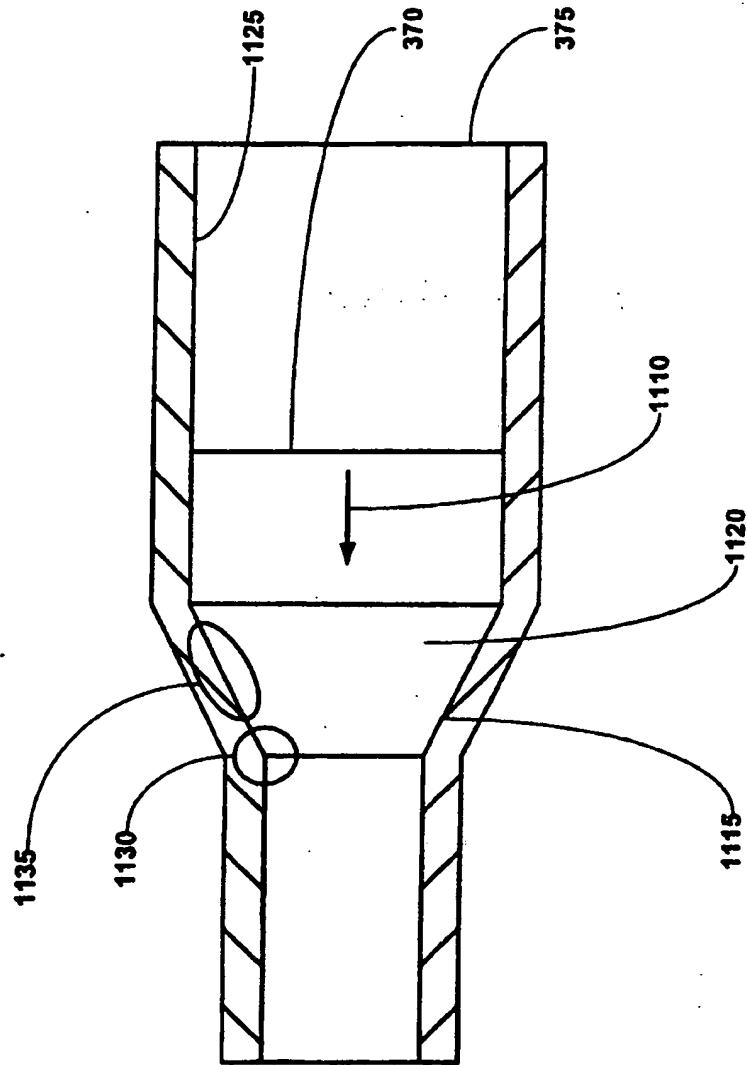
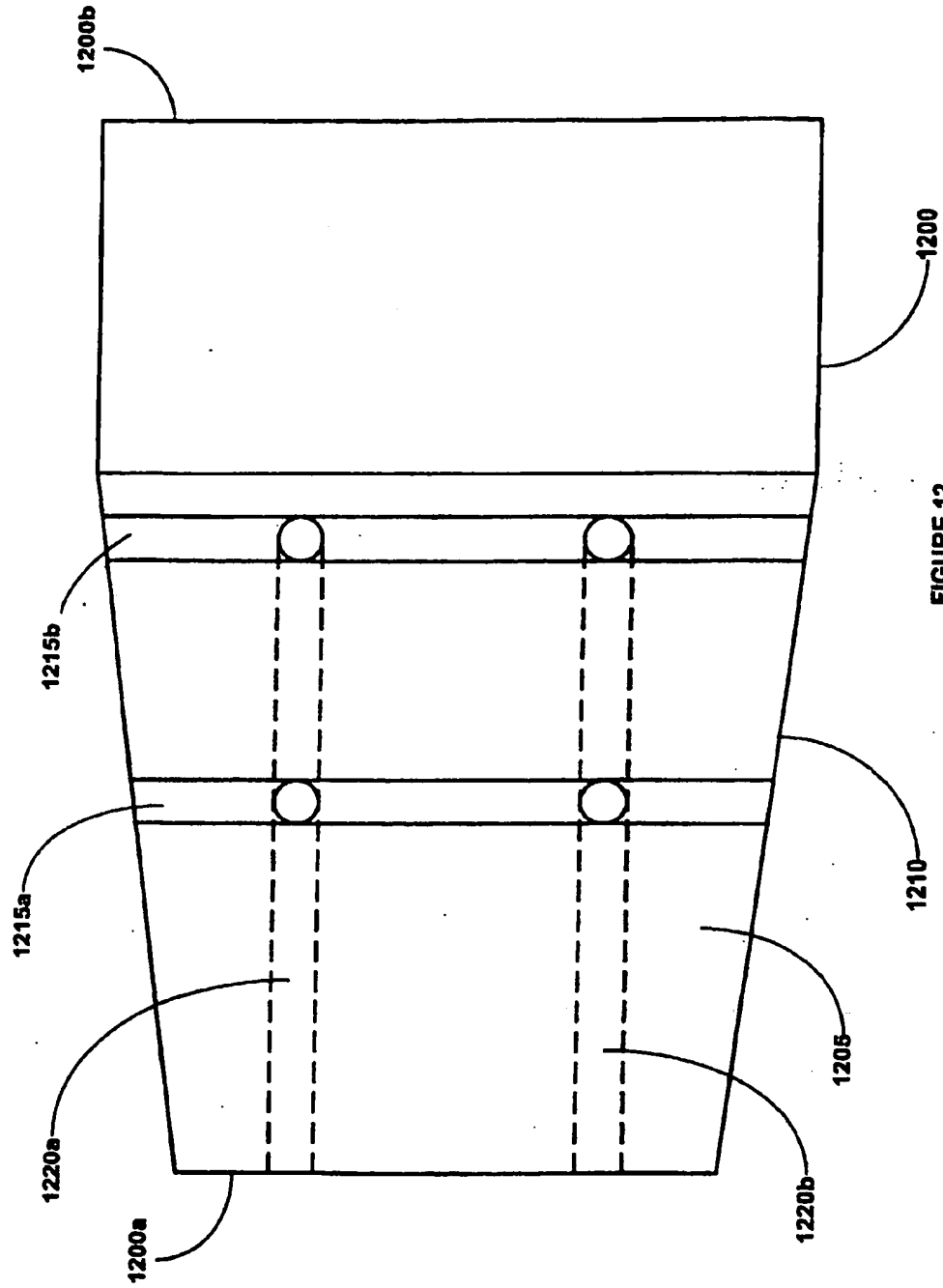


FIGURE 11



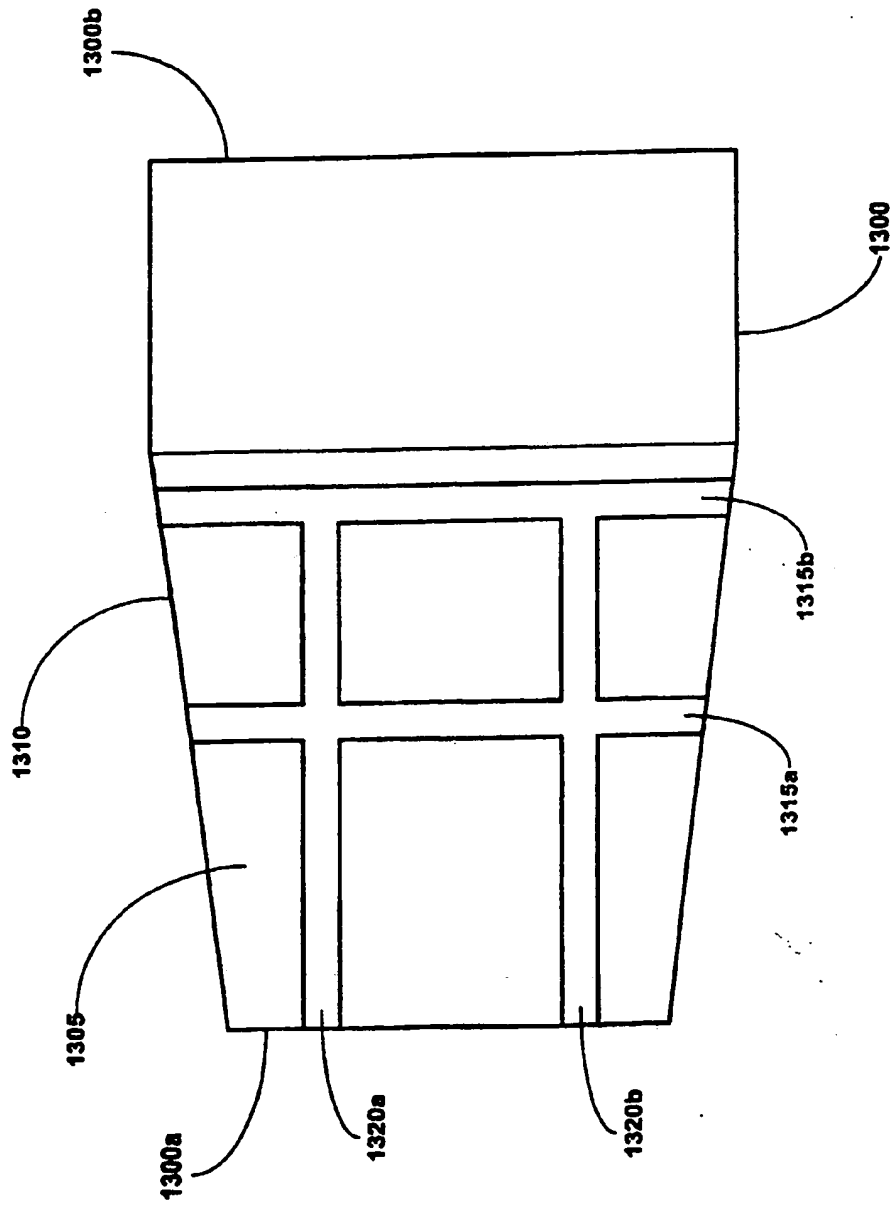


FIGURE 13



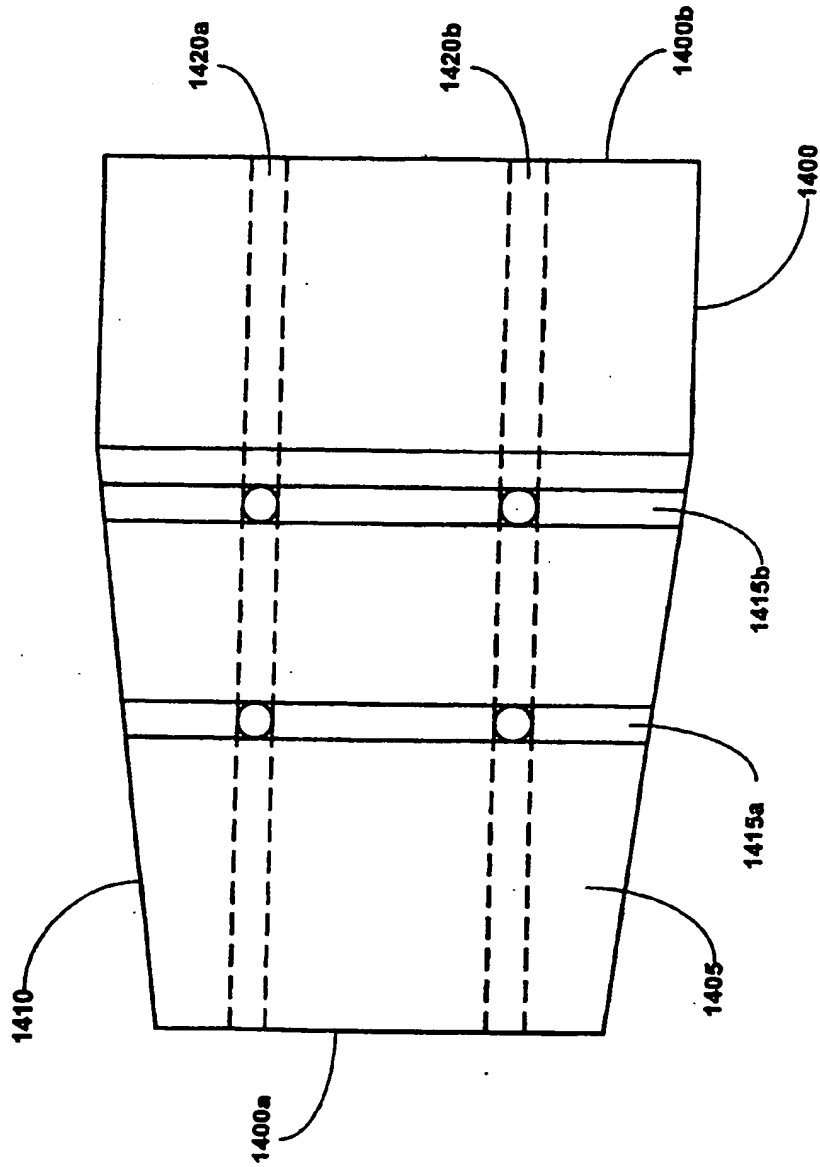


FIGURE 14

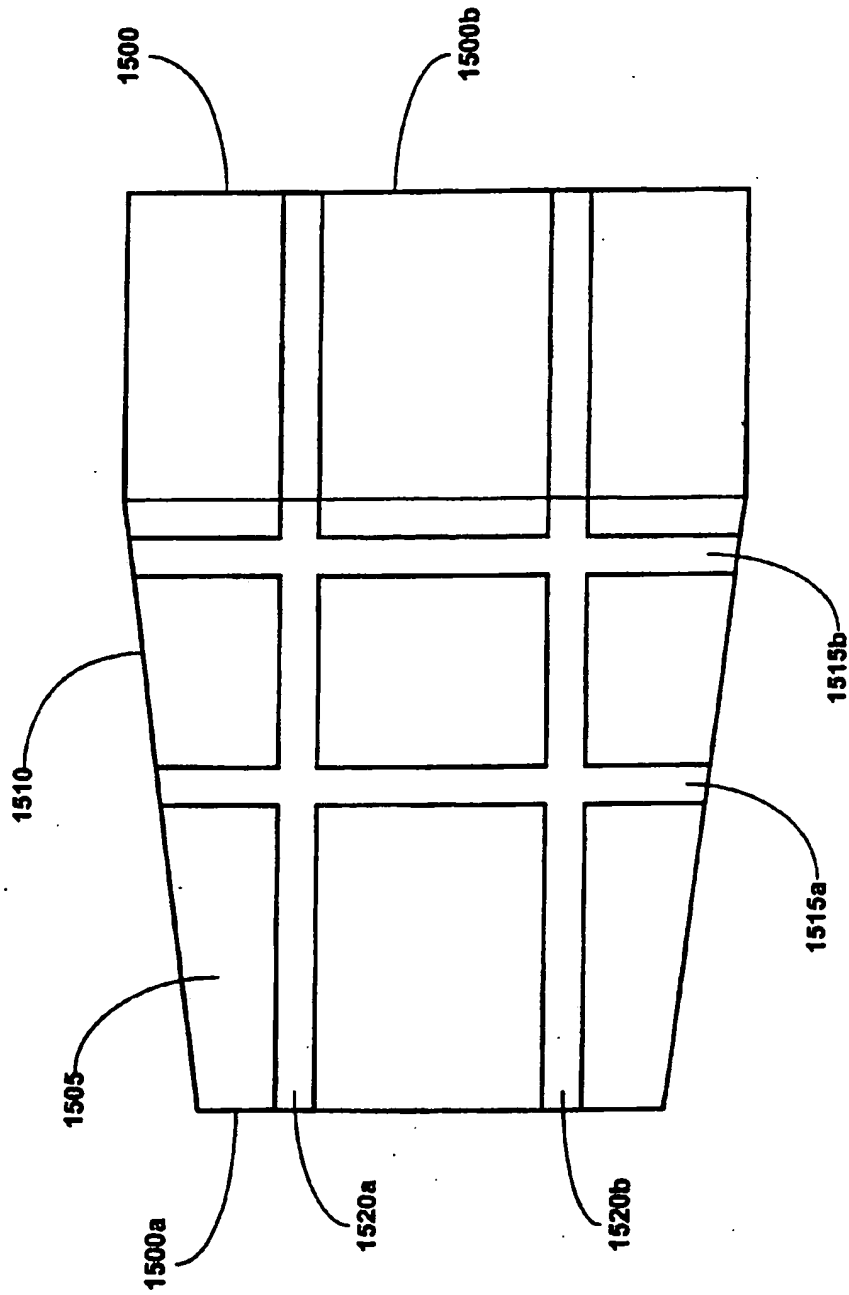


FIGURE 15

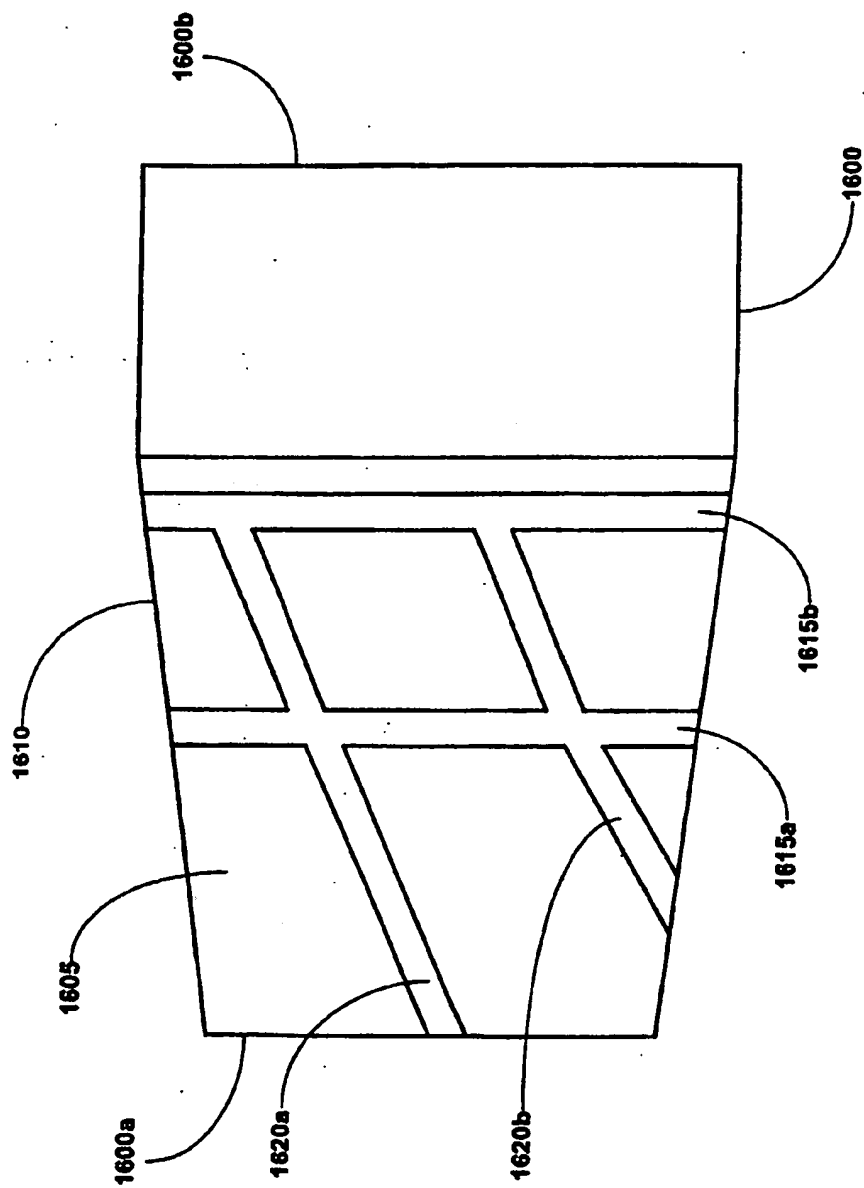


FIGURE 18

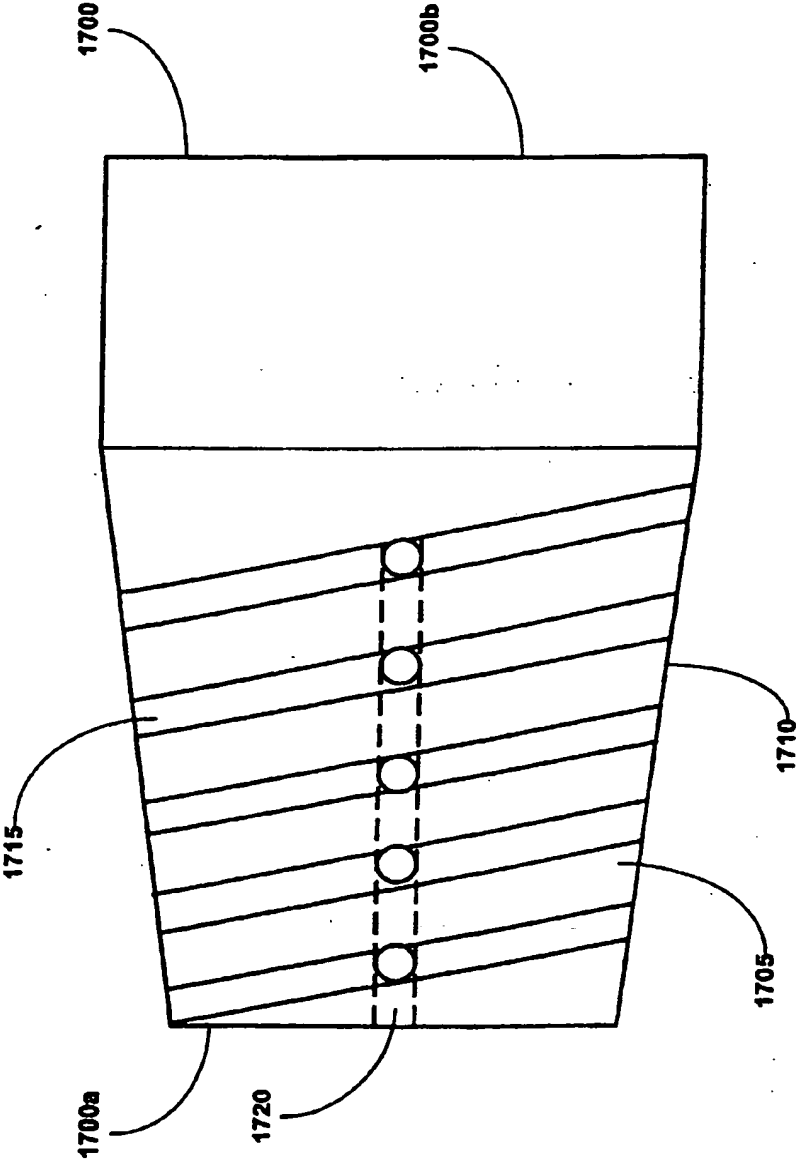


FIGURE 17

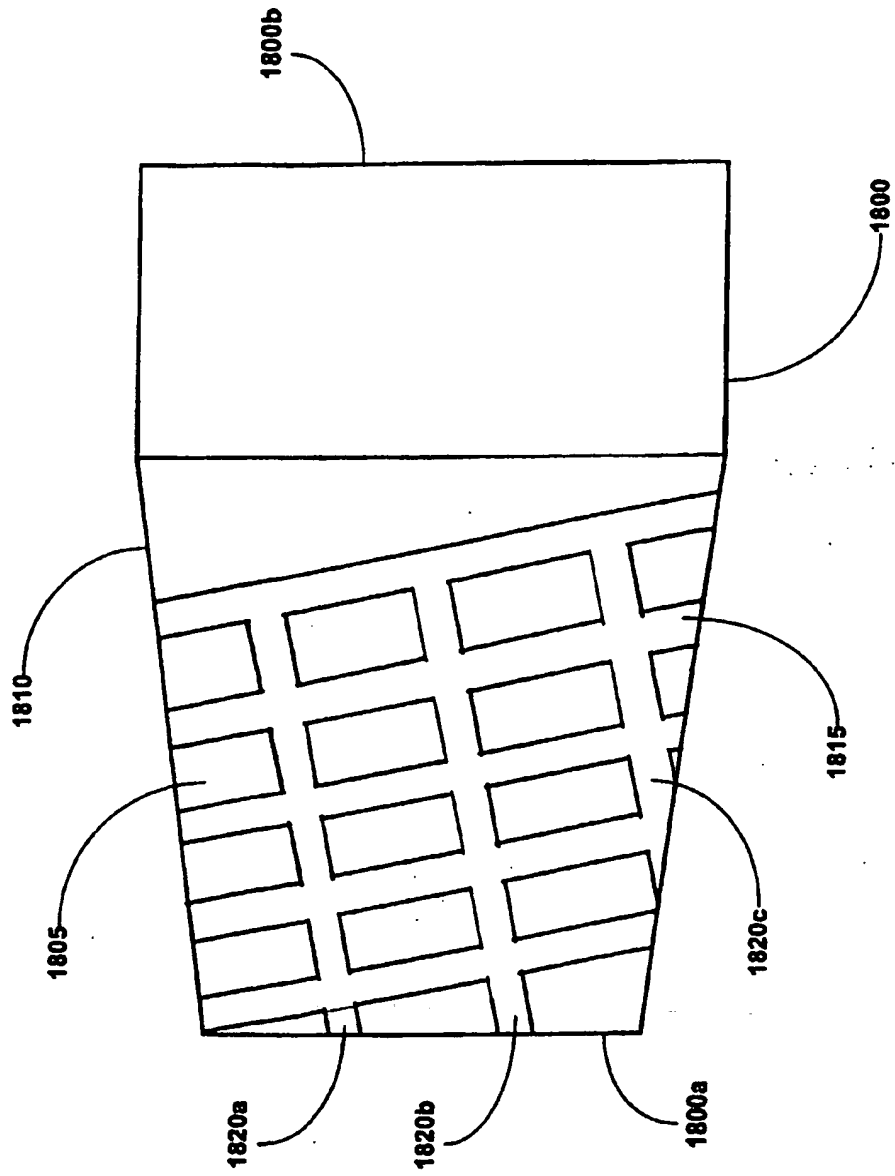


FIGURE 18

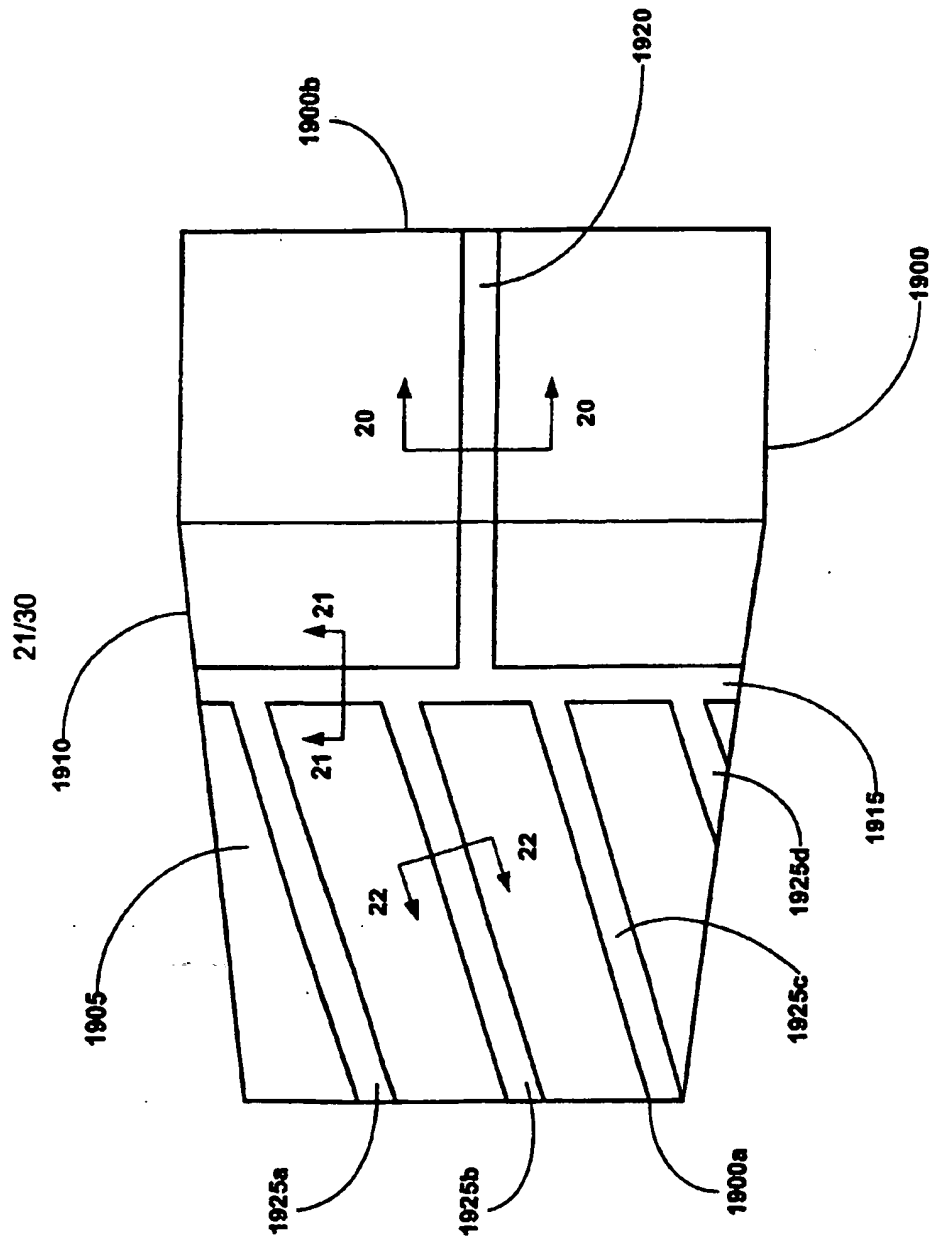


FIGURE 19

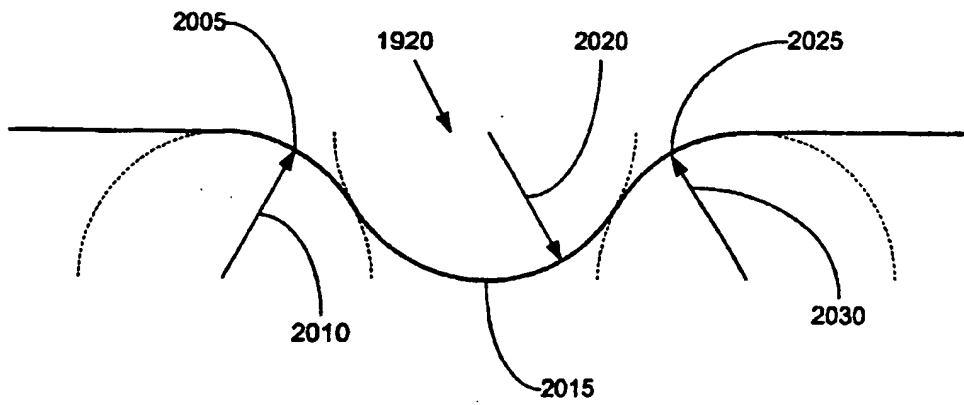


FIGURE 20

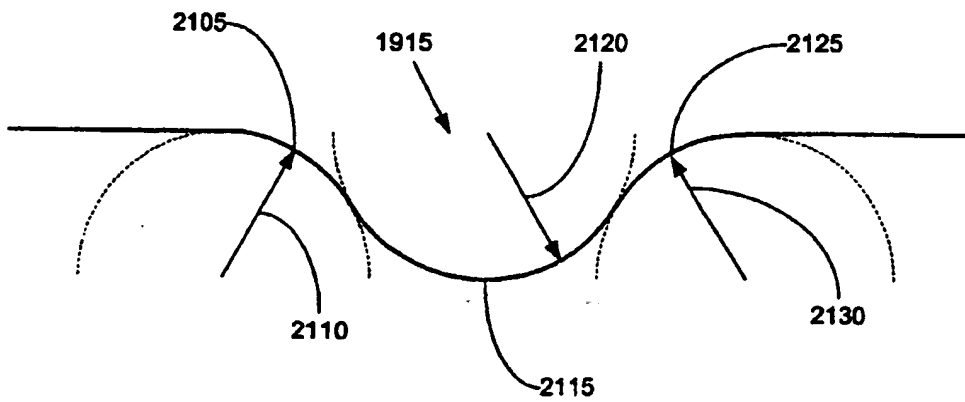


FIGURE 21

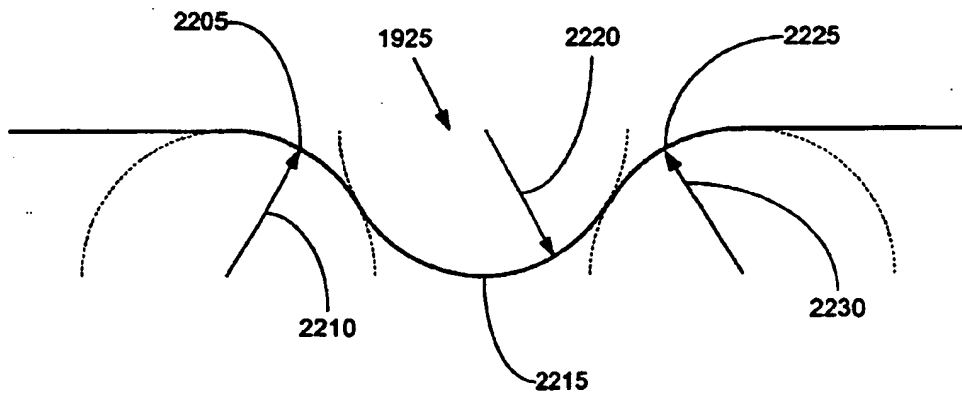


FIGURE 22



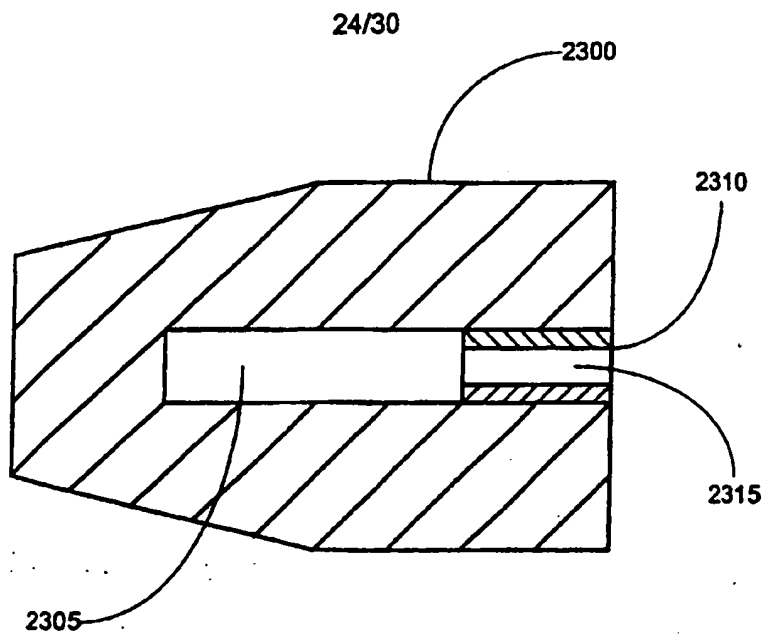


FIGURE 23

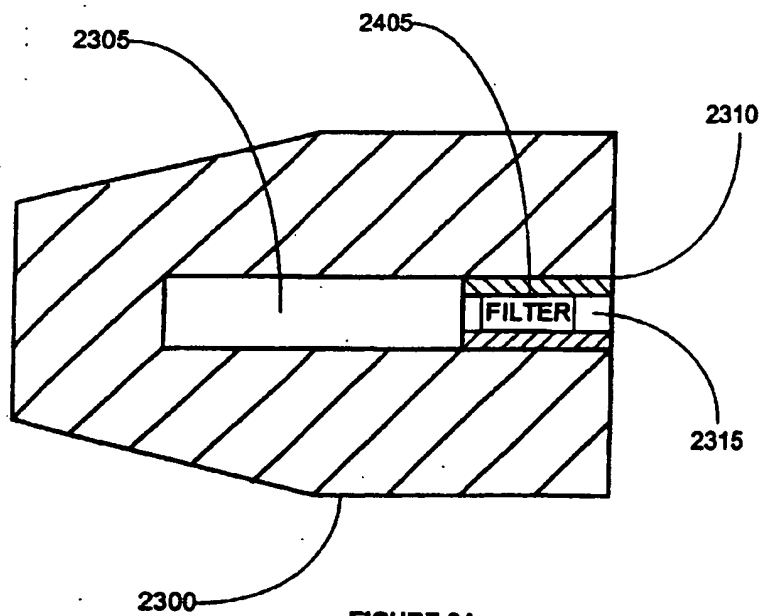


FIGURE 24

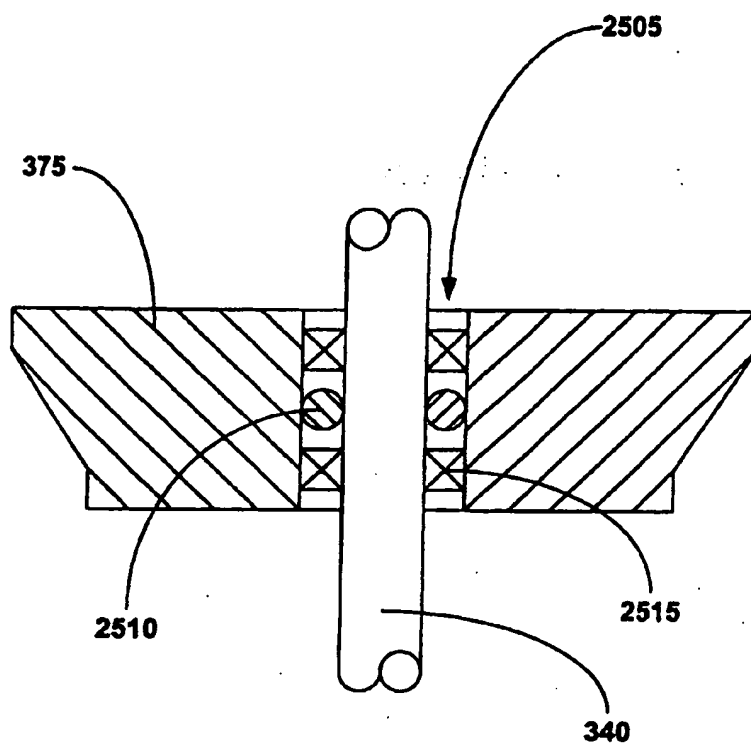


FIGURE 25

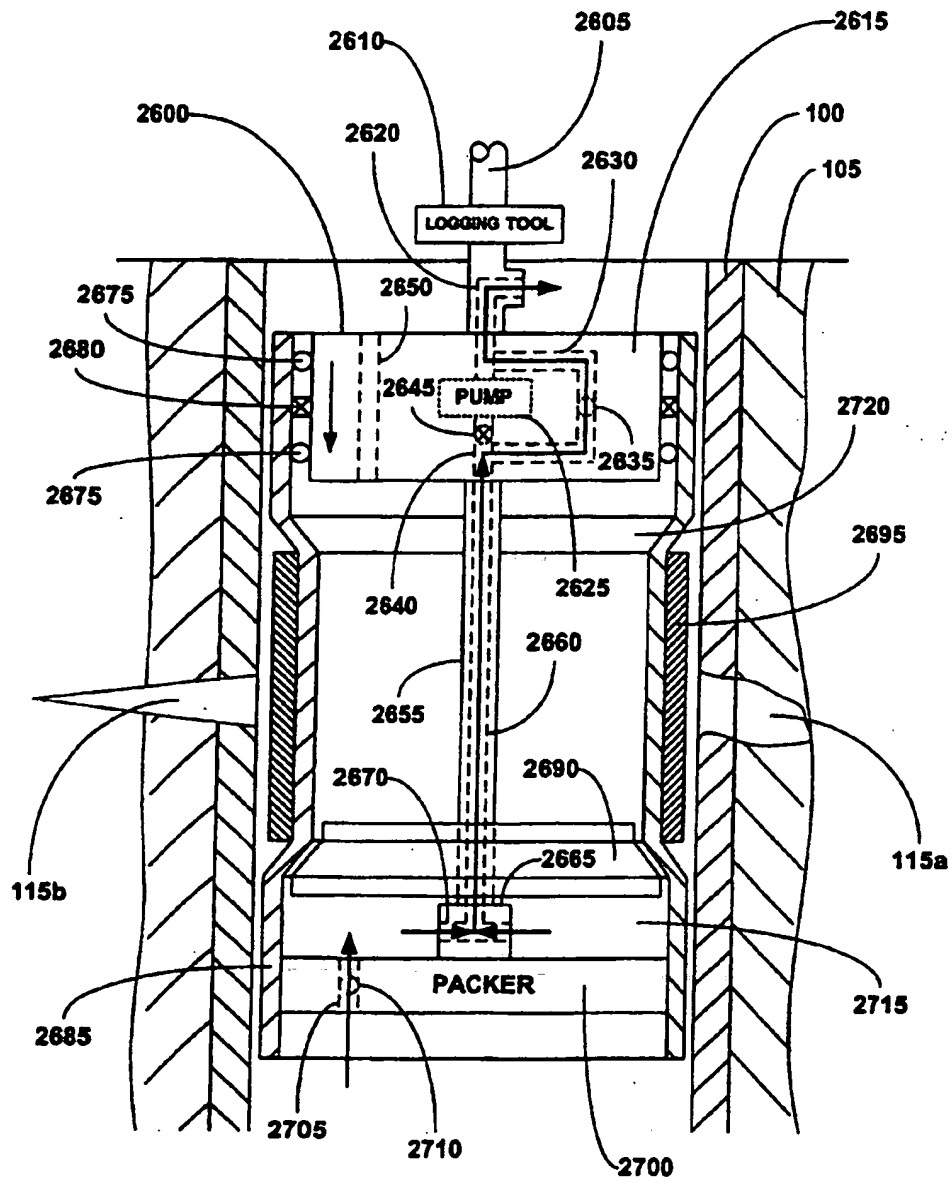


FIGURE 26a

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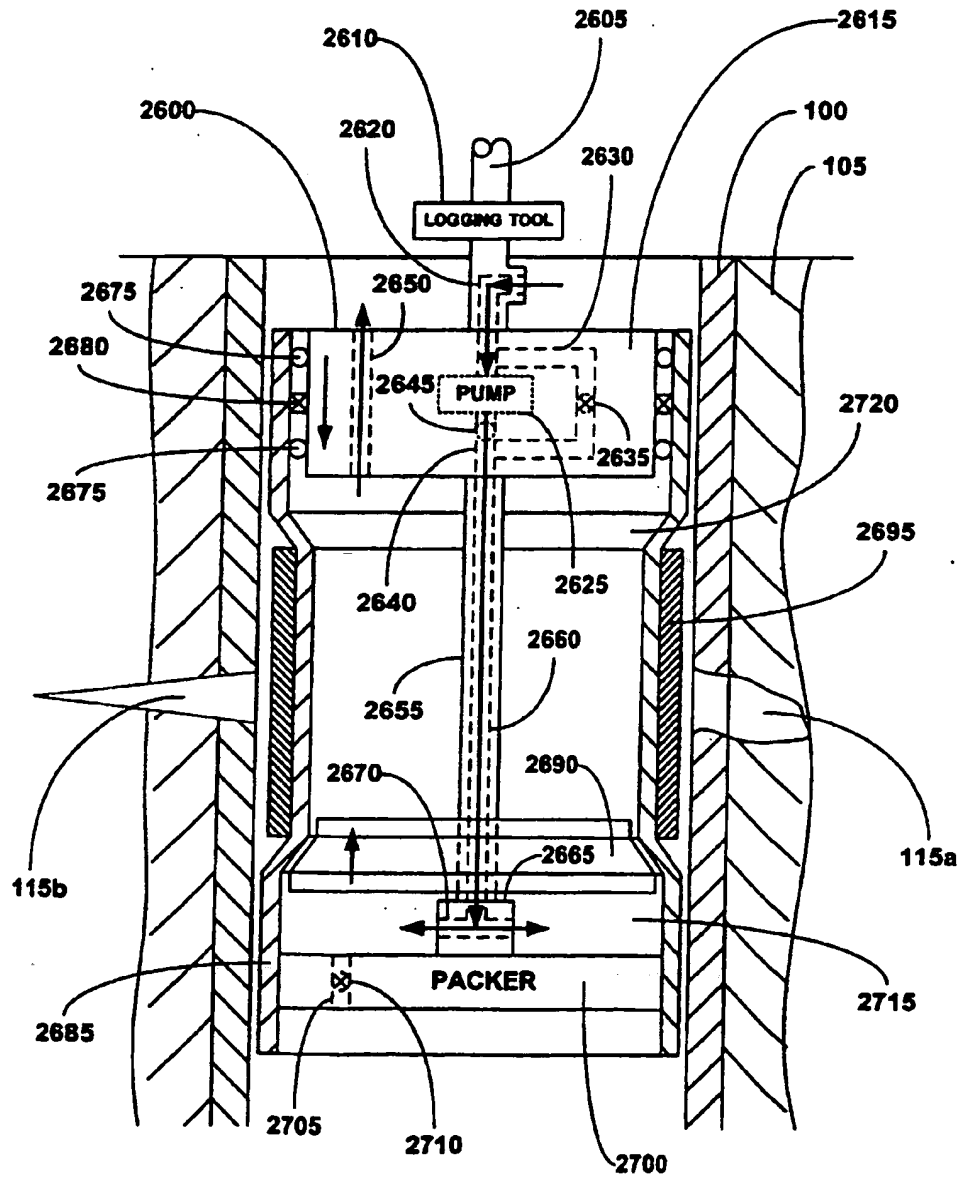


FIGURE 26b

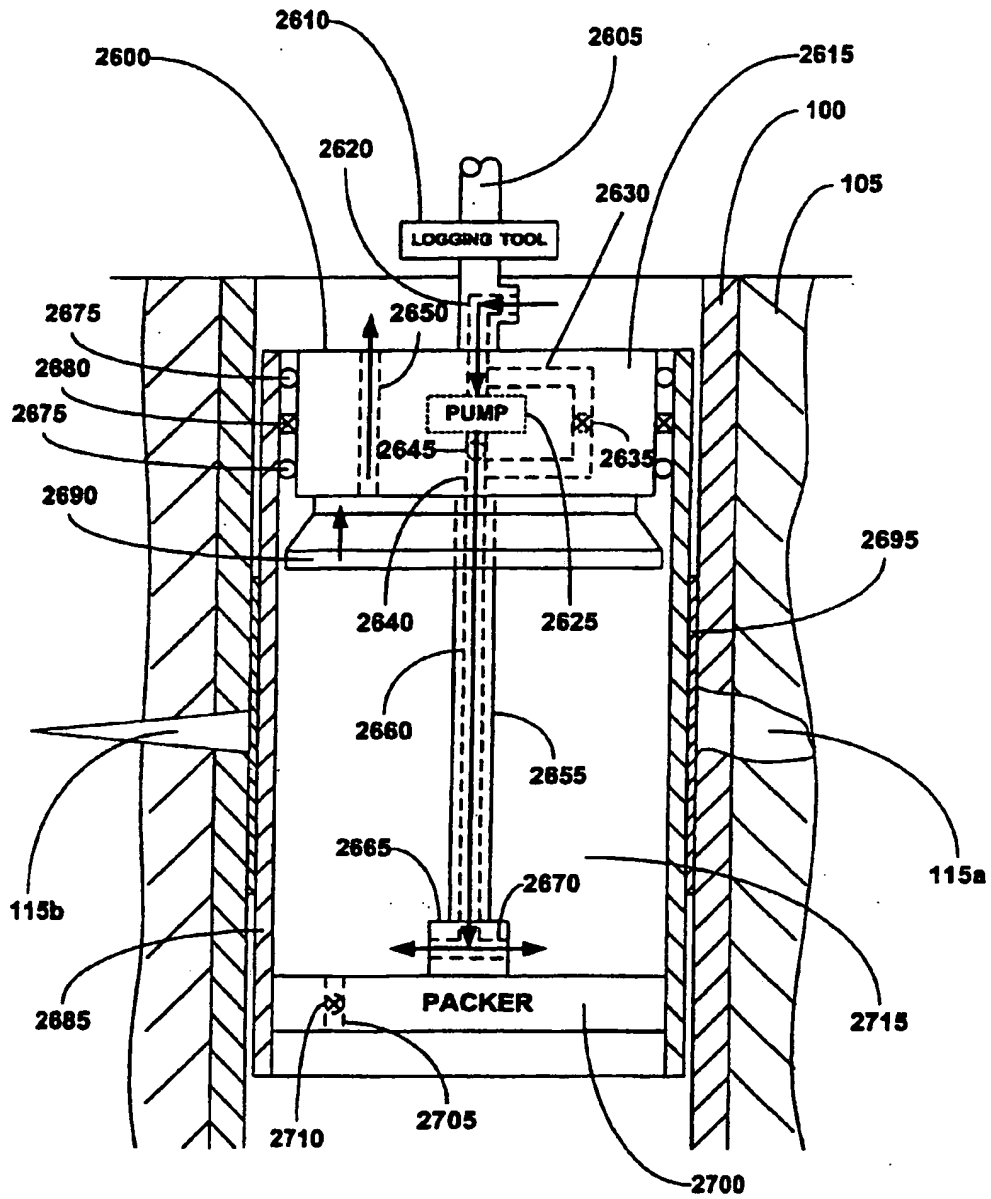


FIGURE 26c

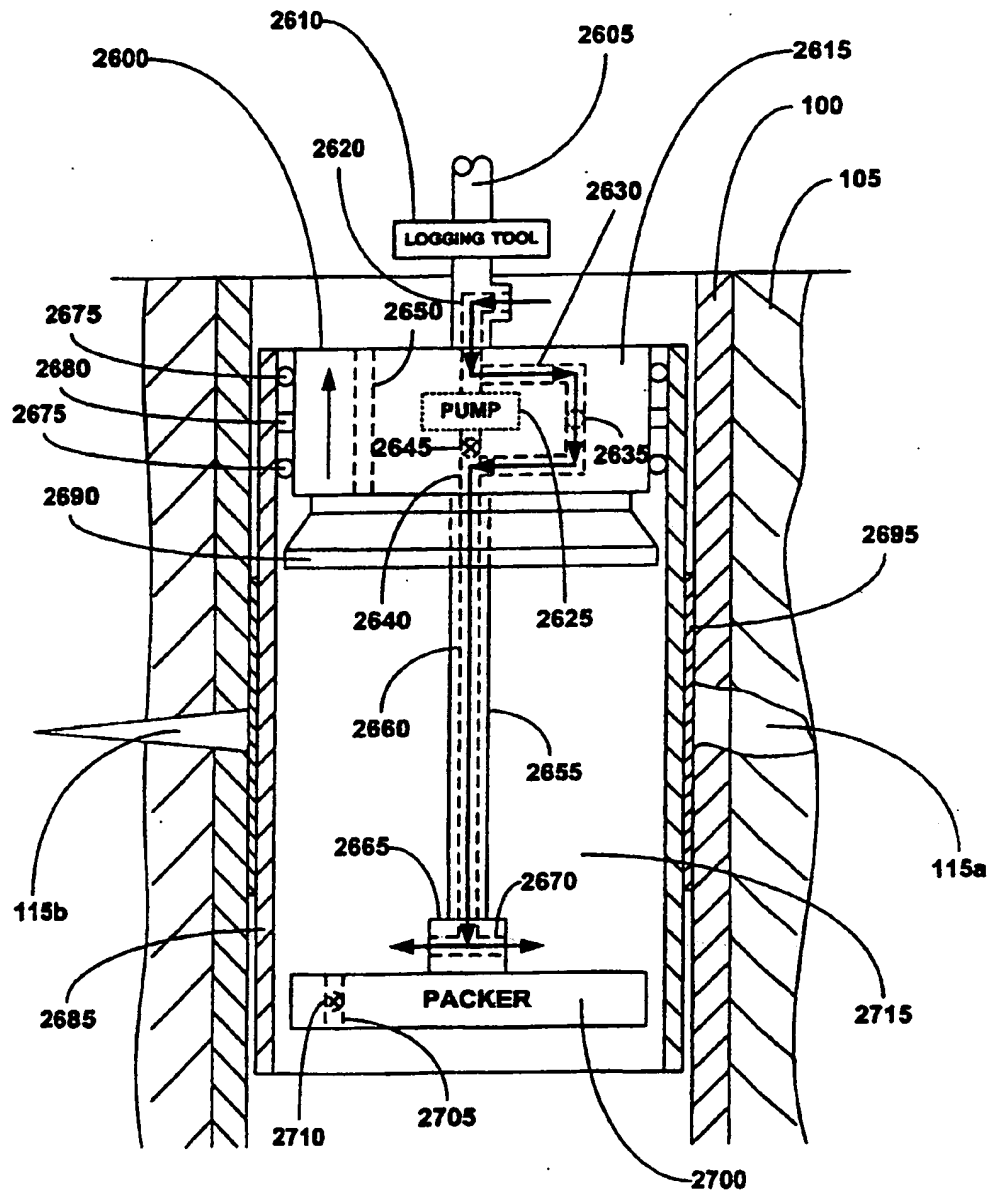


FIGURE 26d

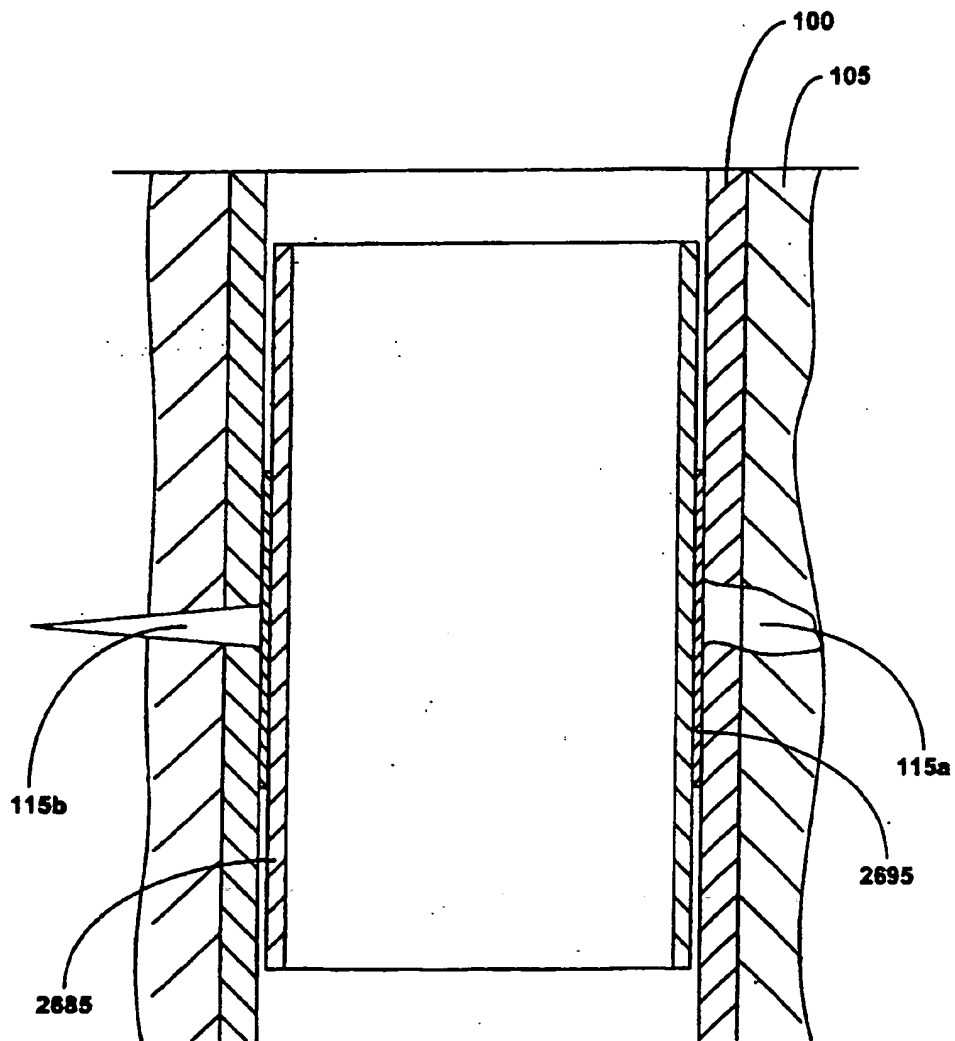


FIGURE 26e

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## WELLBORE CASING REPAIR

### Background of the Invention

This invention relates generally to wellbore casings repair, and in particular  
5 to repair of a wellbore casing that is formed using expandable tubing.

Conventionally, when a wellbore is created, a number of casings are installed  
in the borehole to prevent collapse of the borehole wall and to prevent undesired  
outflow of drilling fluid into the formation or inflow of fluid from the formation into  
the borehole. The borehole is drilled in intervals whereby a casing which is to be  
10 installed in a lower borehole interval is lowered through a previously installed  
casing of an upper borehole interval. As a consequence of this procedure the casing  
of the lower interval is of smaller diameter than the casing of the upper interval.  
Thus, the casings are in a nested arrangement with casing diameters decreasing in  
downward direction. Cement annuli are provided between the outer surfaces of the  
15 casings and the borehole wall to seal the casings from the borehole wall. As a  
consequence of this nested arrangement a relatively large borehole diameter is  
required at the upper part of the wellbore. Such a large borehole diameter involves  
increased costs due to heavy casing handling equipment, large drill bits and  
increased volumes of drilling fluid and drill cuttings. Moreover, increased drilling  
20 rig time is involved due to required cement pumping, cement hardening, required  
equipment changes due to large variations in hole diameters drilled in the course of  
the well, and the large volume of cuttings drilled and removed.

Conventionally, when an opening is formed in the sidewalls of an existing  
wellbore casing, whether through damage to the casing or because of an intentional  
25 perforation of the casing to facilitate production or a fracturing operation, it is often  
necessary to seal off the opening in the existing wellbore casing. Conventional  
methods of sealing off such openings are expensive and unreliable.

The present invention is directed to overcoming one or more of the  
limitations of the existing procedures for forming and repairing wellbores.



### Summary of the Invention

According to a first aspect of the present invention, there is provided an apparatus for repairing a tubular member, comprising:

- a support member;
- 5 an expandable tubular member removably coupled to the support member;
- an expansion cone movably coupled to the support member; and
- a pump coupled to the support member adapted to pressurize a portion of the interior of the expandable tubular member;
- wherein the expandable tubular member includes:
- 10 a first end having a first outer diameter;
- an intermediate portion coupled to the first end having an intermediate outer diameter; and
- a second end having a second outer diameter, and coupled to the intermediate portion;
- 15 wherein the first and second outer diameters are greater than the intermediate outer diameter.

Preferably, the expandable tubular member includes a coating of a lubricant.

Preferably, the expandable tubular member includes a coating of a first component of a lubricant.

- 20 Preferably, the expandable tubular member includes a sealing member coupled to the outer surface of the expandable tubular member.

- Preferably, the first end, second end, and intermediate portion of the expandable tubular member have wall thicknesses  $t_1$ ,  $t_2$ , and  $t_{INT}$  and inside diameters  $D_1$ ,  $D_2$  and  $D_{INT}$ ; and the relationship between the wall thicknesses  $t_1$ ,  $t_2$ , and  $t_{INT}$ , the inside diameters  $D_1$ ,  $D_2$  and  $D_{INT}$ , the inside diameter  $D_{TUBE}$  of the tubular member that the expandable tubular member will be inserted into, and the outside diameter  $D_{cone}$  of the expansion cone is given by the following expression:
- 25

$$D_{TUBE} - 2 * t_1 \geq D_1 \geq \frac{1}{t_1} \left[ (t_1 - t_{INT}) * D_{cone} + t_{INT} * D_{INT} \right]$$



where  $t_1 = t_2$ ; and

$D_1 = D_2$ .

Preferably, the expandable tubular member includes a sealing member coupled to the outside surface of the intermediate portion.

5        Preferably, the expandable tubular member includes:

a first transition portion coupled to the first end and the intermediate portion inclined at a first angle; and

a second transition portion coupled to the second end and the intermediate portion inclined at a second angle;

10       wherein the first and second angles range from 5 to 45 degrees.

Preferably, the expansion cone includes an expansion cone surface having an angle of attack ranging from 10 to 40 degrees.

Preferably, the expansion cone includes:

a first expansion cone surface having a first angle of attack; and

15       a second expansion cone surface having a second angle of attack;

wherein the first angle of attack is greater than the second angle of attack.

Preferably, the expansion cone includes an expansion cone surface having a substantially parabolic profile.

20       Preferably, the expansion cone includes an inclined surface including one or more lubricating grooves.

Preferably, the expansion cone includes one or more internal lubricating passages coupled to each of the lubricating grooves.

According to another aspect of the present invention there is provided an apparatus for repairing a tubular member, comprising:

25       a support member;

an expandable tubular member removably coupled to the support member;

an expansion cone movably coupled to the support member; and

a pump coupled to the support member adapted to pressurize a portion of the interior of the expandable tubular member;

wherein the expansion cone includes an inclined surface including one or more lubricating grooves.

According to another aspect of the present invention there is provided an apparatus for repairing a tubular member, comprising:

- 5 a support member;
- an expandable tubular member removably coupled to the support member;
- an expansion cone movably coupled to the support member; and
- a pump coupled to the support member adapted to pressurize a portion of the interior of the expandable tubular member;

- 10 wherein the expansion cone includes an inclined surface including one or more lubricating grooves; and

wherein the expansion cone includes one or more internal lubricating passages coupled to each of the lubricating grooves.

- 15 According to another aspect of the present invention there is provided an apparatus for repairing a tubular member, comprising:

- a support member;
- an expandable tubular member removably coupled to the support member;
- a tubular expansion cone movably coupled to the support member; and
- a pump coupled to the support member adapted to pressurize a portion of the interior of the expandable tubular member.

- 20 Preferably, the expandable tubular member includes a coating of a lubricant.

Preferably, the expandable tubular member includes a coating of a first component of a lubricant.

- 25 Preferably, the expandable tubular member includes a sealing member coupled to the outer surface of the expandable tubular member.

Preferably, the expandable tubular member includes:

- a first end having a first outer diameter;
- an intermediate portion coupled to the first end having an intermediate outer diameter; and

a second end having a second outer diameter, and coupled to the intermediate portion;

wherein the first and second outer diameters are greater than the intermediate outer diameter.

- 5 Preferably, the first end, second end, and intermediate portion of the expandable tubular member have wall thicknesses  $t_1$ ,  $t_2$ , and  $t_{INT}$  and inside diameters  $D_1$ ,  $D_2$  and  $D_{INT}$ ; and the relationship between the wall thicknesses  $t_1$ ,  $t_2$ , and  $t_{INT}$ , the inside diameters  $D_1$ ,  $D_2$  and  $D_{INT}$ , the inside diameter  $D_{TUBE}$  of the tubular member that the expandable tubular member will be inserted into, and the
- 10 outside diameter  $D_{CONE}$  of the expansion cone is given by the following expression:

$$D_{TUBE} - 2 * t_1 \geq D_1 \geq \frac{1}{t_1} \left[ (t_1 - t_{INT}) * D_{CONE} + t_{INT} * D_{INT} \right]$$

where  $t_1 = t_2$ ; and

$D_1 = D_2$ .

- 15 Preferably, the expandable tubular member includes a sealing member coupled to the outside surface of the intermediate portion.

Preferably, the expandable tubular member includes:

a first transition portion coupled to the first end and the intermediate portion inclined at a first angle; and

- 20 a second transition portion coupled to the second end and the intermediate portion inclined at a second angle;

wherein the first and second angles range from 5 to 45 degrees.

Preferably, the tubular expansion cone includes an expansion cone surface having an angle of attack ranging from 10 to 40 degrees.

- 25 Preferably, the tubular expansion cone includes:

a first expansion cone surface having a first angle of attack; and

a second expansion cone surface having a second angle of attack;

wherein the first angle of attack is greater than the second angle of attack.

Preferably, the tubular expansion cone includes an expansion cone surface having a substantially parabolic profile.

Preferably, the tubular expansion cone includes an inclined surface including one or more lubricating grooves.

- 5        Preferably, the tubular expansion cone includes one or more internal lubricating passages coupled to each of the lubricating grooves.

Preferably, the expandable tubular member includes a sealing member coupled to the outer surface of the expandable tubular member.

Preferably, the expandable tubular member includes:

- 10        a first end having a first outer diameter;  
          an intermediate portion coupled to the first end having an intermediate outer diameter; and  
          a second end having a second outer diameter, and coupled to the intermediate portion;
- 15        wherein the first and second outer diameters are greater than the intermediate outer diameter.

Preferably, the expandable tubular member includes:

- a first transition portion coupled to a first end and an intermediate portion inclined at a first angle; and
- 20        a second transition portion coupled to a second end and the intermediate portion inclined at a second angle;
- wherein the first and second angles range from 5 to 45 degrees.

Preferably, the expansion cone includes an expansion cone surface having an angle of attack ranging from 10 to 40 degrees.

- 25        Preferably, the expansion cone includes:

          a first expansion cone surface having a first angle of attack; and  
          a second expansion cone surface having a second angle of attack;  
          wherein the first angle of attack is greater than the second angle of attack.

- Preferably, the expansion cone includes an expansion cone surface having a
- 30        substantially parabolic profile.

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Preferably, the expandable tubular member includes a sealing member coupled to the outer surface of the expandable tubular member.

Preferably, the expandable tubular member includes:

- a first end having a first outer diameter;
- 5 an intermediate portion coupled to the first end having an intermediate outer diameter; and
- a second end having a second outer diameter, and coupled to the intermediate portion;
- wherein the first and second outer diameters are greater than the intermediate
- 10 outer diameter.

Preferably, the expandable tubular member includes:

- a first transition portion coupled to a first end and an intermediate portion inclined at a first angle; and
- a second transition portion coupled to a second end and the intermediate
- 15 portion inclined at a second angle;
- wherein the first and second angles range from 5 to 45 degrees.

Preferably, the expansion cone includes an expansion cone surface having an angle of attack ranging from 10 to 40 degrees.

Preferably, the expansion cone includes:

- 20 a first expansion cone surface having a first angle of attack; and
- a second expansion cone surface having a second angle of attack;
- wherein the first angle of attack is greater than the second angle of attack.

Preferably, the expansion cone includes an expansion cone surface having a substantially parabolic profile.

25

#### Brief Description of the Drawings

FIG. 1 is a fragmentary cross-sectional view of a wellbore casing including one or more openings.

- FIG. 2 is a flow chart illustration of a method for repairing the wellbore
- 30 casing of FIG. 1.

FIG. 3a is a fragmentary cross-sectional view of the placement of a repair apparatus within the wellbore casing of FIG. 1 wherein the expandable tubular member of the apparatus is positioned opposite the openings in the wellbore casing.

FIG. 3b is a fragmentary cross-sectional view of the radial expansion of the expandable tubular of the apparatus of FIG. 3a.

FIG. 3c is a fragmentary cross-sectional view of the completion of the radial expansion of the expandable tubular of the apparatus of FIG. 3b.

FIG. 3d is a fragmentary cross-sectional view of the removal of the repair apparatus from the repaired wellbore casing of FIG. 3c.

FIG. 3e is a fragmentary cross-sectional view of the repaired wellbore casing of FIG. 3d.

FIG. 4 is a cross-sectional illustration of the expandable tubular of the apparatus of FIG. 3a.

FIG. 5 is a flow chart illustration of a method for fabricating the expandable tubular of the apparatus of FIG. 3a.

FIG. 6 is a fragmentary cross-sectional illustration of the expandable tubular of FIG. 4.

FIG. 7 is a fragmentary cross-sectional illustration of an expansion cone expanding a tubular member.

FIG. 8 is a graphical illustration of the relationship between propagation pressure and the angle of attack of the expansion cone.

FIG. 9 is an illustration of an expansion cone optimally adapted to radially expand the expandable tubular member of FIG. 4.

FIG. 10 is an illustration of an expansion cone optimally adapted to radially expand the expandable tubular member of FIG. 4.

FIG. 11 is a fragmentary cross-sectional illustration of the lubrication of the interface between an expansion cone and a tubular member during the radial expansion process.

FIG. 12 is an illustration of an expansion cone including a system for lubricating the interface between the expansion cone and a tubular member during the radial expansion of the tubular member.

5 FIG. 13 is an illustration of an expansion cone including a system for lubricating the interface between the expansion cone and a tubular member during the radial expansion of the tubular member.

FIG. 14 is an illustration of an expansion cone including a system for lubricating the interface between the expansion cone and a tubular member during the radial expansion of the tubular member.

10 FIG. 15 is an illustration of an expansion cone including a system for lubricating the interface between the expansion cone and a tubular member during the radial expansion of the tubular member.

FIG. 16 is an illustration of an expansion cone including a system for lubricating the interface between the expansion cone and a tubular member during  
15 the radial expansion of the tubular member.

FIG. 17 is an illustration of an expansion cone including a system for lubricating the interface between the expansion cone and a tubular member during the radial expansion of the tubular member.

FIG. 18 is an illustration of an expansion cone including a system for  
20 lubricating the interface between the expansion cone and a tubular member during the radial expansion of the tubular member.

FIG. 19 is an illustration of an expansion cone including a system for lubricating the interface between the expansion cone and a tubular member during the radial expansion of the tubular member.

25 FIG. 20 is a cross-sectional illustration of the first axial groove of the expansion cone of FIG. 19.

FIG. 21 is a cross-sectional illustration of the circumferential groove of the expansion cone of FIG. 19.

FIG. 22 is a cross-sectional illustration of one of the second axial grooves of  
30 the expansion cone of FIG. 19.



FIG. 23 is a cross sectional illustration of an expansion cone including internal flow passages having inserts for adjusting the flow of lubricant fluids.

FIG. 24 is a cross sectional illustration of the expansion cone of FIG. 23 further including an insert having a filter for filtering out foreign materials from the  
5 lubricant fluids.

FIG. 25 is a fragmentary cross sectional illustration of the expansion cone of the repair apparatus of FIG. 3a.

FIG. 26a is a fragmentary cross-sectional view of the placement of a repair apparatus within the wellbore casing of FIG. 1 wherein the expandable tubular  
10 member of the apparatus is positioned opposite the openings in the wellbore casing.

FIG. 26b is a fragmentary cross-sectional view of the radial expansion of the expandable tubular of the apparatus of FIG. 26a.

FIG. 26c is a fragmentary cross-sectional view of the completion of the radial expansion of the expandable tubular of the apparatus of FIG. 26b.

FIG. 26d is a fragmentary cross-sectional view of the removal of the repair  
15 apparatus from the repaired wellbore casing of FIG. 26c.

FIG. 26e is a fragmentary cross-sectional view of the repaired wellbore casing of FIG. 26d.

## 20 Detailed Description

Referring initially to FIG. 1, a wellbore casing 100 having an outer annular layer 105 of a sealing material is positioned within a subterranean formation 110. The wellbore casing 100 may be positioned in any orientation from vertical to horizontal. The wellbore casing 100 further includes one or more openings 115a  
25 and 115b. The openings 115 may, for example, be the result of: defects in the wellbore casing 100, intentional perforations of the casing to facilitate production, thin walled sections of casing caused by drilling and/or wireline wear, or fracturing operations. As will be recognized by persons having ordinary skill in the art, such openings 115 in a wellbore 100 can seriously adversely impact the subsequent  
30 production of oil and gas from the subterranean formation 110 unless they are sealed

off. More generally, the wellbore casing 115 may include thin walled sections that need cladding in order to prevent a catastrophic failure.

Referring to FIG. 2, a method 200 for repairing a defect in a wellbore casing using a repair apparatus having a logging tool, a pump, an expansion cone, and an expandable tubular member includes the steps of: (1) positioning the repair apparatus within the wellbore casing in step 205; (2) locating the defect in the wellbore casing using the logging tool of the repair apparatus in step 210; (3) positioning the expandable tubular member in opposition to the defect in the wellbore casing in step 215; and (4) radially expanding the expandable tubular member into intimate contact with the wellbore casing by pressurizing a portion of the expandable tubular member using the pump and extruding the expandable tubular member off of the expansion cone in step 220. In this manner, defects in a wellbore casing are repaired by a compact and self-contained repair apparatus that is positioned downhole. More generally, the repair apparatus is used to repair defects in wellbore casings, pipelines, and structural supports.

305 is further preferably adapted to convey electrical power and communication signals to the logging tool 310, the pump 325, and the locking member 365.

The logging tool 310 is preferably coupled to the first support member 305. The logging tool 310 is preferably adapted to detect defects in the wellbore casing 100. The logging tool 310 may be any number of conventional commercially available logging tools suitable for detecting defects in wellbore casings, pipelines, or structural supports. The logging tool 310 is a CAST logging tool, available from Halliburton<sup>(RTM)</sup> Energy Services in order to optimally provide detection of defects in the wellbore casing 100. The logging tool 310 is contained within the housing 315 in order to provide an repair apparatus 300 that is rugged and compact.

The housing 315 is preferably coupled to the first support member 305, the second support member 340, the sealing members 360, and the locking member 365. The housing 315 is preferably releasably coupled to the tubular member 370. The housing 315 is further preferably adapted to contain and/or support the logging tool 310 and the pump 325.

The first fluid conduit 320 is preferably fluidically coupled to the inlet of the pump 325 and the exterior region above the housing 315. The first fluid conduit 320 may be contained within the first support member 305 and the housing 315. The first fluid conduit 320 is preferably adapted to convey fluidic materials such as, for example, drilling muds, water, and lubricants at operating pressures and flow rates ranging from about 0 to 12,000 psi and 0 to 500 gallons/minute in order to optimally propagate the expansion cone 375.

The pump 325 is fluidically coupled to the first fluid conduit 320 and the second fluid conduit 330. The pump 325 is further preferably contained within and supported by the housing 315. Alternatively, the pump 325 may be positioned above the housing 315. The pump 325 is preferably adapted to convey fluidic materials from the first fluid conduit 320 to the second fluid conduit 330 at operating pressures and flow rates ranging from about 0 to 12,000 psi and 0 to 500 gallons/minute in order to optimally provide the operating pressure for propagating the expansion cone 375. The pump 325 may be any number of conventional

commercially available pumps. The pump 325 is a flow control pump out section for dirty fluids, available from Halliburton<sup>(RTM)</sup> Energy Services in order to optimally provide the operating pressures and flow rates for propagating the expansion cone 375. The pump 325 is preferably adapted to pressurize an interior portion 385 of the expandable tubular member 370 to operating pressures ranging from about 0 to 12,000 psi.

The second fluid conduit 330 is fluidically coupled to the outlet of the pump 325 and the interior portion 385 of the expandable tubular member 370. The second fluid conduit 330 is further preferably contained within the housing 315. The second fluid conduit 330 is preferably adapted to convey fluidic materials such as, for example, drilling muds, water, and lubricants at operating pressures and flow rates ranging from about 0 to 12,000 psi and 0 to 500 gallons/minute in order to optimally propagate the expansion cone 375.

The third fluid conduit 335 is fluidically coupled to the exterior region above the housing 315 and the interior portion 385 of the expandable tubular member 370. The third fluid conduit 335 is further preferably contained within the housing 315. The third fluid conduit 330 is preferably adapted to convey fluidic materials such as, for example, drilling muds, water, and lubricants at operating pressures and flow rates ranging from about 0 to 12,000 psi and 0 to 500 gallons/minute in order to optimally propagate the expansion cone 375.

The second support member 340 is coupled to the housing 315 and the third support member 350. The second support member 340 is further preferably movably and sealingly coupled to the expansion cone 375. The second support member 340 preferably has a substantially annular cross section in order to provide one or more conduits for conveying fluidic materials. The second support member 340 is centrally positioned within the expandable tubular member 370.

The fourth fluid conduit 345 is fluidically coupled to the third fluid conduit 335 and the fifth fluid conduit 355. The fourth fluid conduit 345 is further preferably contained within the second support member 340. The fourth fluid conduit 345 is preferably adapted to convey fluidic materials such as, for example,

drilling muds, water, and lubricants at operating pressures and flow rates ranging from about 0 to 12,000 psi and 0 to 500 gallons/minute in order to optimally propagate the expansion cone 375.

The third support member 350 is coupled to the second support member 340.

- 5 The third support member 350 is further preferably adapted to support the expansion cone 375. The third support member 350 preferably has a substantially annular cross section in order to provide one or more conduits for conveying fluidic materials.

- 10 The fifth fluid conduit 355 is fluidically coupled to the fourth fluid conduit 345 and a portion 390 of the expandable tubular member 375 below the expansion cone 375. The fifth fluid conduit 355 is further preferably contained within the third support member 350. The fifth fluid conduit 355 is preferably adapted to convey fluidic materials such as, for example, drilling muds, water, and lubricants at operating pressures and flow rates ranging from about 0 to 12,000 psi and 0 to 500  
15 gallons/minute in order to optimally propagate the expansion cone 375.

- The sealing members 360 are preferably coupled to the housing 315. The sealing members 360 are preferably adapted to seal the interface between the exterior surface of the housing 315 and the interior surface of the expandable tubular member 370. In this manner, the interior portion 385 of the expandable tubular  
20 member 375 is fluidically isolated from the exterior region above the housing 315. The sealing members 360 may be any number of conventional commercially available sealing members. The sealing members 360 are conventional O-ring sealing members available from various commercial suppliers in order to optimally provide a high pressure seal.

- 25 The locking member 365 is preferably coupled to the housing 315. The locking member 365 is further preferably releasably coupled to the expandable tubular member 370. In this manner, the housing 365 is controllably coupled to the expandable tubular member 370. In this manner, the housing 365 is preferably released from the expandable tubular member 370 upon the completion of the radial  
30 expansion of the expandable tubular member 370. The locking member 365 may



be any number of conventional commercially available releasable locking members. The locking member 365 is an electrically releasable locking member in order to optimally provide an easily retrievable running expansion system.

5 The locking member 365 is replaced by or supplemented by one or more conventional shear pins in order to provide an alternative means of controllably releasing the housing 315 from the expandable tubular member 370.

The expandable tubular member 370 is releasably coupled to the locking member 365. The expandable tubular member 370 is preferably adapted to be radially expanded by the axial displacement of the expansion cone 375.

10 As illustrated in FIG. 4, the expandable tubular member 370 includes a tubular body 405 having an interior region 410, an exterior surface 415, a first end 420, an intermediate portion 425, and a second end 430. The tubular member 370 further preferably includes the sealing member 380.

The tubular body 405 of the tubular member 370 preferably has a substantially annular cross section. The tubular body 405 may be fabricated from any number of conventional commercially available materials such as, for example, Oilfield Country Tubular Goods (OCTG), 13 chromium steel, 4140 steel, or automotive grade steel tubing/casing, or L83, J55, or P110 API casing. The tubular body 405 of the tubular member 370 is further provided substantially as disclosed in  
20 one or more of the following co-pending U.S. patent applications:

Provisional Patent Application Number	Attorney Docket No.	Filing Date
60/108,558	25791.9	11-16-1998
60/111,293	25791.3	12-7-1998
60/119,611	25791.8	2-11-1999
60/121,702	25791.7	2-25-1999
60/121,841	25791.12	2-26-1999
60/121,907	25791.16	2-26-1999

Provisional Patent Application Number	Attorney Docket No.	Filing Date
60/124,042	25791.11	3-11-1999
60/131,106	25791.23	4-26-1999
60/137,998	25791.17	6-7-1999
60/143,039	25791.26	7-9-1999
60/146,203	25791.25	7-29-1999
60/154,047	25791.29	9-16-1999
60/159,082	25791.34	10-12-1999
60/159,039	25791.36	10-12-1999
60/159,033	25791.37	10-12-1999

The interior region 410 of the tubular body 405 preferably has a substantially circular cross section. The interior region 410 of the tubular body 405 preferably includes a first inside diameter  $D_1$ , an intermediate inside diameter  $D_{INT}$ , and a  
5 second inside diameter  $D_2$ . The first and second inside diameters,  $D_1$  and  $D_2$ , are substantially equal. The first and second inside diameters,  $D_1$  and  $D_2$ , are greater than the intermediate inside diameter  $D_{INT}$ .

The first end 420 of the tubular body 405 is coupled to the intermediate portion 425 of the tubular body 405. The exterior surface of the first end 420 of the  
10 tubular body 405 preferably further includes a protective coating fabricated from tungsten carbide, or other similar wear resistant materials in order to protect the first end 420 of the tubular body 405 during placement of the repair apparatus 300 within the wellbore casing 100. The outside diameter of the first end 420 of the tubular body 405 is greater than the outside diameter of the intermediate portion 425 of the  
15 tubular body 405. In this manner, the sealing member 380 is optimally protected during placement of the tubular member 370 within the wellbore casing 100. The outside diameter of the first end 420 of the tubular body 405 is substantially equal to the outside diameter of the second end 430 of the tubular body 405. In this manner,

the sealing member 380 is optimally protected during placement of the tubular member 370 within the wellbore casing 100. The outside diameter of the first end 420 of the tubular member 370 is adapted to permit insertion of the tubular member 370 into the typical range of wellbore casings. The first end 420 of the tubular member 370 includes a wall thickness  $t_1$ .

The intermediate portion 425 of the tubular body 405 is coupled to the first end 420 of the tubular body 405 and the second end 430 of the tubular body 405. The intermediate portion 425 of the tubular body 405 preferably includes the sealing member 380. The outside diameter of the intermediate portion 425 of the tubular body 405 is less than the outside diameter of the first and second ends, 420 and 430, of the tubular body 405. In this manner, the sealing member 380 is optimally protected during placement of the tubular member 370 within the wellbore casing 100. The outside diameter of the intermediate portion 425 of the tubular body 405 ranges from about 75% to 98% of the outside diameters of the first and second ends, 420 and 430, in order to optimally protect the sealing member 380 during placement of the tubular member 370 within the wellbore casing 100. The intermediate portion 425 of the tubular body 405 includes a wall thickness  $t_{INT}$ .

The second end 430 of the tubular body 405 is coupled to the intermediate portion 425 of the tubular body 405. The exterior surface of the second end 430 of the tubular body 405 preferably further includes a protective coating fabricated from a wear resistant material such as, for example, tungsten carbide in order to protect the second end 430 of the tubular body 405 during placement of the repair apparatus 300 within the wellbore casing 100. The outside diameter of the second end 430 of the tubular body 405 is greater than the outside diameter of the intermediate portion 425 of the tubular body 405. In this manner, the sealing member 380 is optimally protected during placement of the tubular member 370 within a wellbore casing 100. The outside diameter of the second end 430 of the tubular body 405 is substantially equal to the outside diameter of the first end 420 of the tubular body 405. In this manner, the sealing member 380 is optimally protected during placement of the tubular member 370 within the wellbore casing 100. The outside diameter of the



second end 430 of the tubular member 370 is adapted to permit insertion of the tubular member 370 into the typical range of wellbore casings. The second end 430 of the tubular member 370 includes a wall thickness  $t_2$ .

5 The wall thicknesses  $t_1$  and  $t_2$  are substantially equal in order to provide substantially equal burst strength for the first and second ends, 420 and 430, of the tubular member 370. The wall thicknesses  $t_1$  and  $t_2$  are both greater than the wall thickness  $t_{INT}$  in order to optimally match the burst strength of the first and second ends, 420 and 430, of the tubular member 370 with the intermediate portion 425 of the tubular member 370.

10 The sealing member 380 is preferably coupled to the outer surface of the intermediate portion 425 of the tubular body 405. The sealing member 380 preferably seals the interface between the intermediate portion 425 of the tubular body 405 and interior surface of the wellbore casing 100 after radial expansion of the intermediate portion 425 of the tubular body 405. The sealing member 380  
15 preferably has a substantially annular cross section. The outside diameter of the sealing member 380 is preferably selected to be less than the outside diameters of the first and second ends, 420 and 430, of the tubular body 405 in order to optimally protect the sealing member 380 during placement of the tubular member 370 within the typical range of wellbore casings 100. The sealing member 380 may be  
20 fabricated from any number of conventional commercially available materials such as, for example, thermoset or thermoplastic polymers. The sealing member 380 is fabricated from thermoset polymers in order to optimally seal the interface between the radially expanded intermediate portion 425 of the tubular body 405 and the wellbore casing 100.

25 During placement of the tubular member 370 within the wellbore casing 100, the protective coatings provided on the exterior surfaces of the first and second ends, 420 and 430, of the tubular body 405 prevent abrasion with the interior surface of the wellbore casing 100. After radial expansion of the tubular body 405, the sealing member 380 seals the interface between the outside surface of the intermediate  
30 portions 425 of the tubular body 405 of the tubular member 370 and the inside

surface of the wellbore casing 100. During placement of the tubular member 370 within the wellbore casing 100, the sealing member 380 is preferably protected from contact with the interior walls of the wellbore casing 100 by the recessed outer surface profile of the tubular member 370.

5        The tubular body 405 of the tubular member 370 further includes first and second transition portions, 435 and 440, coupled between the first and second ends, 420 and 430, and the intermediate portion 425 of the tubular body 405. The first and second transition portions, 435 and 440, are inclined at an angle,  $\alpha$ , relative to the longitudinal direction ranging from about 0 to 30 degrees in order to optimally  
10   facilitate the radial expansion of the tubular member 370. The first and second transition portions, 435 and 440, provide a smooth transition between the first and second ends, 420 and 430, and the intermediate portion 425, of the tubular body 405 of the tubular member 370 in order to minimize stress concentrations.

Referring to FIG. 5, The tubular member 370 is formed by a process 500 that  
15   includes the steps of: (1) expanding both ends of the tubular body 405 in step 505; (2) stress relieving both radially expanded ends of the tubular body 405 in step 510; and (3) putting a sealing material on the outside diameter of the non-expanded intermediate portion 425 of the tubular body 405 in step 515. The process 500 further includes the step of putting layers of protective coatings onto the exterior  
20   surfaces of the radially expanded ends, 420 and 430, of the tubular body 405.

In steps 505 and 510, both ends, 420 and 430, of the tubular body 405 are radially expanded using conventional radial expansion methods, and then both ends, 420 and 430, of the tubular body 405 are stress relieved. The radially expanded ends, 420 and 430, of the tubular body 405 include interior diameters  $D_1$  and  $D_2$ .  
25   The interior diameters  $D_1$  and  $D_2$  are substantially equal in order to provide a burst strength that is substantially equal. The ratio of the interior diameters  $D_1$  and  $D_2$  to the interior diameter  $D_{INT}$  of the tubular body 405 ranges from about 100% to 120% in order to optimally provide a tubular member for subsequent radial expansion.

The relationship between the wall thicknesses  $t_1$ ,  $t_2$ , and  $t_{INT}$  of the tubular  
30   body 405; the inside diameters  $D_1$ ,  $D_2$  and  $D_{INT}$  of the tubular body 405; the inside

diameter  $D_{\text{wellbore}}$  of the wellbore casing 100 that the tubular body 405 will be inserted into; and the outside diameter  $D_{\text{cone}}$  of the expansion cone 375 that will be used to radially expand the tubular body 405 within the wellbore casing 100 is given by the following expression:

$$D_{\text{wellbore}} - 2 * t_1 \geq D_1 \geq \frac{1}{t_1} [(t_1 - t_{\text{INT}}) * D_{\text{cone}} + t_{\text{INT}} * D_{\text{INT}}^5] \quad (1)$$

where  $t_1 = t_2$ ; and

$$D_1 = D_2.$$

By satisfying the relationship given in equation (1), the expansion forces placed upon the tubular body 405 during the subsequent radial expansion process are substantially equalized. More generally, the relationship given in equation (1) may be used to calculate the optimal geometry for the tubular body 405 for subsequent radial expansion of the tubular body 405 for fabricating and/or repairing a wellbore casing, a pipeline, or a structural support.

In step 515, the sealing member 380 is then applied onto the outside diameter of the non-expanded intermediate portion 425 of the tubular body 405. The sealing member 380 may be applied to the outside diameter of the non-expanded intermediate portion 425 of the tubular body 405 using any number of conventional commercially available methods. The sealing member 380 is applied to the outside diameter of the intermediate portion 425 of the tubular body 405 using commercially available chemical and temperature resistant adhesive bonding.

As illustrated in FIG. 6, the interior surface of the tubular body 405 of the tubular member 370 further includes a coating 605 of a lubricant. The coating 605 of lubricant may be applied using any number of conventional methods such as, for example, dipping, spraying, sputter coating or electrostatic deposition. The coating 605 of lubricant is chemically, mechanically, and/or adhesively bonded to the interior surface of the tubular body 405 of the tubular member 370 in order to optimally provide a durable and consistent lubricating effect. The force that bonds the lubricant to the interior surface of the tubular body 405 of the tubular member 370 is greater than the shear force applied during the radial expansion process.

The coating 605 of lubricant is applied to the interior surface of the tubular body 405 of the tubular member 370 by first applying a phenolic primer to the interior surface of the tubular body 405 of the tubular member 370, and then bonding the coating 605 of lubricant to the phenolic primer using an antifriction  
5 paste including the coating 605 of lubricant carried within an epoxy resin. The antifriction paste includes, by weight, 40-80% epoxy resin, 15-30% molybdenum disulfide, 10-15% graphite, 5-10% aluminum, 5-10% copper, 8-15% aluminosilicate, and 5-10% polyethylenepolyamine. The antifriction paste is provided substantially as disclosed in U.S. Patent No. 4,329,238, the disclosure of which is incorporate  
10 herein by reference.

The coating 605 of lubricant may be any number of conventional commercially available lubricants such as, for example, metallic soaps or zinc phosphates. The coating 605 of lubricant includes C-Lube-10, C-Phos-52, C-Phos-58-M, and/or C-Phos-58-R in order to optimally provide a coating of lubricant. The  
15 coating 605 of lubricant provides a sliding coefficient of friction less than about 0.20 in order to optimally reduce the force required to radially expand the tubular member 370 using the expansion cone 375.

The coating 605 includes a first part of a lubricant. The first part of the lubricant forms a first part of a metallic soap. The first part of the lubricant coating  
20 includes zinc phosphate. The second part of the lubricant is circulated within a fluidic carrier that is circulated into contact with the coating 605 of the first part of the lubricant during the radial expansion of the tubular member 370. The first and second parts of the lubricant react to form a lubricating layer between the interior surface of the tubular body 405 of the tubular member 370 and the exterior surface  
25 of the expansion cone 375 during the radial expansion process. In this manner, a lubricating layer is optimally provided in the exact concentration, exactly when and where it is needed. Furthermore, because the second part of the lubricant is circulated in a carrier fluid, the dynamic interface between the interior surface of the tubular body 405 of the tubular members 370 and the exterior surface of the  
30 expansion cone 375 is also preferably provided with hydrodynamic lubrication. The

first and second parts of the lubricant react to form a metallic soap. The second part of the lubricant is sodium stearate.

The expansion cone 375 is movably coupled to the second support member 340. The expansion cone 375 is preferably adapted to be axially displaced upon the pressurization of the interior region 385 of the expandable tubular member 370. The expansion cone 375 is further preferably adapted to radially expand the expandable tubular member 370.

As illustrated in FIG. 7, the expansion cone 375 includes a conical outer surface 705 for radially expanding the tubular member 370 having an angle of attack  $\alpha$ . As illustrated in FIG. 8, the angle of attack  $\alpha$  ranges from about 10 to 40 degrees in order to minimize the required operating pressure of the interior portion 385 during the radial expansion process.

Referring to FIG. 9, an expansion cone 900 for use in the repair apparatus 300 includes a front end 905, a rear end 910, and a radial expansion section 915. When the expansion cone 900 is displaced in the longitudinal direction relative to the tubular member 370, the interaction of the exterior surface of the radial expansion section 915 with the interior surface of the tubular member 370 causes the tubular member 370 to expand in the radial direction.

The radial expansion section 915 preferably includes a leading radial expansion section 920 and a trailing radial expansion section 925. The leading and trailing radial expansion sections, 920 and 925, have substantially conical outer surfaces. The leading and trailing radial expansion sections, 920 and 925, have corresponding angles of attack,  $\alpha_1$  and  $\alpha_2$ . The angle of attack  $\alpha_1$  of the leading radial expansion section 920 is greater than the angle of attack  $\alpha_2$  of the trailing radial expansion section 925 in order to optimize the radial expansion of the tubular member 370. More generally, the radial expansion section 915 may include one or more intermediate radial expansion sections positioned between the leading and trailing radial expansion sections, 920 and 925, wherein the corresponding angles of attack  $\alpha$  increase in stepwise fashion from the leading radial expansion section 920 to the trailing radial expansion section 925.

Referring to FIG. 10, an expansion cone 1000 for use in the repair apparatus 300 includes a front end 1005, a rear end 1010, and a radial expansion section 1015. When the expansion cone 1000 is displaced in the longitudinal direction relative to the tubular member 370, the interaction of the exterior surface of the radial expansion section 1015 with the interior surface of the tubular member 370 causes the tubular member 370 to expand in the radial direction.

The radial expansion section 1015 preferably includes an outer surface 1020 having a substantially parabolic outer profile. In this manner, the outer surface 1020 provides an angle of attack that constantly decreases from a maximum at the front end 1005 of the expansion cone 1000 to a minimum at the rear end 1010 of the expansion cone 1000. The parabolic outer profile of the outer surface 1020 may be formed using a plurality of adjacent discrete conical sections and/or using a continuous curved surface. In this manner, the area of the outer surface 1020 adjacent to the front end 1005 of the expansion cone 1000 optimally radially overexpands the intermediate portion 425 of the tubular body 405 of the tubular member 370, while the area of the outer surface 1020 adjacent to the rear end 1010 of the expansion cone 1000 optimally radially overexpands the pre-expanded first and second ends, 420 and 430, of the tubular body 405 of the tubular member 370. The parabolic profile of the outer surface 1020 is selected to provide an angle of attack that ranges from about 8 to 20 degrees in the vicinity of the front end 1005 of the expansion cone 1000 and an angle of attack in the vicinity of the rear end 1010 of the expansion cone 1000 from about 4 to 15 degrees.

Referring to FIG. 11, the lubrication of the interface between the expansion cone 370 and the tubular member 375 during the radial expansion process will now be described. As illustrated in FIG. 31, during the radial expansion process, an expansion cone 370 radially expands the tubular member 375 by moving in an axial direction 1110 relative to the tubular member 375. The interface between the outer surface 1115 of the tapered conical portion 1120 of the expansion cone 370 and the inner surface 1125 of the tubular member 375 includes a leading edge portion 1130 and a trailing edge portion 1135.

During the radial expansion process, the leading and trailing edge portions, 1130 and 1135, are preferably lubricated by the presence of the coating 605 of lubricant. During the radial expansion process, the leading edge portion 5025 is further lubricated by the presence of lubricating fluids provided ahead of the expansion cone 370. However, because the radial clearance between the expansion cone 370 and the tubular member 375 in the trailing edge portion 1135 during the radial expansion process is typically extremely small, and the operating contact pressures between the tubular member 375 and the expansion cone 370 are extremely high, the quantity of lubricating fluid provided to the trailing edge portion 1135 is typically greatly reduced. In typical radial expansion operations, this reduction in the flow of lubricating fluids in the trailing edge portion 1135 increases the forces required to radially expand the tubular member 375.

Referring to FIG. 12, An expansion cone 1200 is used in the repair apparatus 300 that includes a front end 1200a, a rear end 1200b, a tapered portion 1205 having an outer surface 1210, one or more circumferential grooves 1215a and 1215b, and one more internal flow passages 1220a and 1220b.

The circumferential grooves 1215 are fluidically coupled to the internal flow passages 1220. In this manner, during the radial expansion process, lubricating fluids are transmitted from the area ahead of the front 1200a of the expansion cone 1200 into the circumferential grooves 1215. Thus, the trailing edge portion of the interface between the expansion cone 1200 and the tubular member 370 is provided with an increased supply of lubricant, thereby reducing the amount of force required to radially expand the tubular member 370. The lubricating fluids are injected into the internal flow passages 1220 using a fluid conduit that is coupled to the tapered end 1205 of the expansion cone 1200. Alternatively, lubricating fluids are provided for the internal flow passages 1220 using a supply of lubricating fluids provided adjacent to the front 1200a of the expansion cone 1200.

The expansion cone 1200 includes a plurality of circumferential grooves 1215. The cross sectional area of the circumferential grooves 1215 range from about  $2 \times 10^{-4} \text{ in}^2$  to  $5 \times 10^{-2} \text{ in}^2$  in order to optimally provide lubrication to the trailing

edge portion of the interface between the expansion cone 1200 and the tubular member 370 during the radial expansion process. The expansion cone 1200 includes circumferential grooves 1215 concentrated about the axial midpoint of the tapered portion 1205 in order to optimally provide lubrication to the trailing edge portion of the interface between the expansion cone 1200 and a tubular member during the radial expansion process. The circumferential grooves 1215 are equally spaced along the trailing edge portion of the expansion cone 1200 in order to optimally provide lubrication to the trailing edge portion of the interface between the expansion cone 1200 and the tubular member 370 during the radial expansion process.

The expansion cone 1200 includes a plurality of flow passages 1220 coupled to each of the circumferential grooves 1215. The cross-sectional area of the flow passages 1220 ranges from about  $2 \times 10^{-4} \text{ in}^2$  to  $5 \times 10^{-2} \text{ in}^2$  in order to optimally provide lubrication to the trailing edge portion of the interface between the expansion cone 1200 and the tubular member 370 during the radial expansion process. The cross sectional area of the circumferential grooves 1215 is greater than the cross sectional area of the flow passage 1220 in order to minimize resistance to fluid flow.

Referring to FIG. 13, an expansion cone 1300 is used in the repair apparatus 300 that includes a front end 1300a and a rear end 1300b, includes a tapered portion 1305 having an outer surface 1310, one or more circumferential grooves 1315a and 1315b, and one or more axial grooves 1320a and 1320b.

The circumferential grooves 1315 are fluidically coupled to the axial grooves 1320. In this manner, during the radial expansion process, lubricating fluids are transmitted from the area ahead of the front 1300a of the expansion cone 1300 into the circumferential grooves 1315. Thus, the trailing edge portion of the interface between the expansion cone 1300 and the tubular member 370 is provided with an increased supply of lubricant, thereby reducing the amount of force required to radially expand the tubular member 370. The axial grooves 1320 are provided with lubricating fluid using a supply of lubricating fluid positioned proximate the front



end 1300a of the expansion cone 1300. The circumferential grooves 1315 are concentrated about the axial midpoint of the tapered portion 1305 of the expansion cone 1300 in order to optimally provide lubrication to the trailing edge portion of the interface between the expansion cone 1300 and the tubular member 370 during the radial expansion process. The circumferential grooves 1315 are equally spaced along the trailing edge portion of the expansion cone 1300 in order to optimally provide lubrication to the trailing edge portion of the interface between the expansion cone 1300 and the tubular member 370 during the radial expansion process.

10 The expansion cone 1300 includes a plurality of circumferential grooves 1315. The cross sectional area of the circumferential grooves 1315 range from about  $2 \times 10^{-4} \text{ in}^2$  to  $5 \times 10^{-2} \text{ in}^2$  in order to optimally provide lubrication to the trailing edge portion of the interface between the expansion cone 1300 and the tubular member 370 during the radial expansion process.

15 The expansion cone 1300 includes a plurality of axial grooves 1320 coupled to each of the circumferential grooves 1315. The cross sectional area of the axial grooves 1320 ranges from about  $2 \times 10^{-4} \text{ in}^2$  to  $5 \times 10^{-2} \text{ in}^2$  in order to optimally provide lubrication to the trailing edge portion of the interface between the expansion cone 1300 and the tubular member 370 during the radial expansion process. The cross sectional area of the circumferential grooves 1315 is greater than the cross sectional area of the axial grooves 1320 in order to minimize resistance to fluid flow. The axial grooves 1320 are spaced apart in the circumferential direction by at least about 3 inches in order to optimally provide lubrication during the radial expansion process.

25 Referring to FIG. 14, an expansion cone 1400 is used in the repair apparatus 300 that includes a front end 1400a and a rear end 1400b, includes a tapered portion 1405 having an outer surface 1410, one or more circumferential grooves 1415a and 1415b, and one or more internal flow passages 1420a and 1420b.

30 The circumferential grooves 1415 are fluidically coupled to the internal flow passages 1420. In this manner, during the radial expansion process, lubricating

fluids are transmitted from the areas in front of the front 1400a and/or behind the rear 1400b of the expansion cone 1400 into the circumferential grooves 1415. Thus, the trailing edge portion of the interface between the expansion cone 1400 and the tubular member 370 is provided with an increased supply of lubricant, thereby  
 5 reducing the amount of force required to radially expand the tubular member 370. Furthermore, the lubricating fluids also preferably pass to the area in front of the expansion cone 1400. In this manner, the area adjacent to the front 1400a of the expansion cone 1400 is cleaned of foreign materials. The lubricating fluids are injected into the internal flow passages 1420 by pressurizing the area behind the rear  
 10 1400b of the expansion cone 1400 during the radial expansion process.

The expansion cone 1400 includes a plurality of circumferential grooves 1415. The cross sectional area of the circumferential grooves 1415 ranges from about  $2 \times 10^{-4} \text{ in}^2$  to  $5 \times 10^{-2} \text{ in}^2$  respectively, in order to optimally provide lubrication to the trailing edge portion of the interface between the expansion cone 1400 and the  
 15 tubular member 370 during the radial expansion process. The expansion cone 1400 includes circumferential grooves 1415 that are concentrated about the axial midpoint of the tapered portion 1405 in order to optimally provide lubrication to the trailing edge portion of the interface between the expansion cone 1400 and the tubular member 370 during the radial expansion process. The circumferential grooves 1415  
 20 are equally spaced along the trailing edge portion of the expansion cone 1400 in order to optimally provide lubrication to the trailing edge portion of the interface between the expansion cone 1400 and the tubular member 370 during the radial expansion process.

The expansion cone 1400 includes a plurality of flow passages 1420 coupled  
 25 to each of the circumferential grooves 1415. The flow passages 1420 fluidically couple the front end 1400a and the rear end 1400b of the expansion cone 1400. The cross-sectional area of the flow passages 1420 ranges from about  $2 \times 10^{-4} \text{ in}^2$  to  $5 \times 10^{-2} \text{ in}^2$  in order to optimally provide lubrication to the trailing edge portion of the interface between the expansion cone 1400 and the tubular member 370 during the  
 30 radial expansion process. The cross sectional area of the circumferential grooves

1415 is greater than the cross-sectional area of the flow passages 1420 in order to minimize resistance to fluid flow.

Referring to FIG. 15, an expansion cone 1500 is used in the apparatus that includes a front end 1500a and a rear end 1500b, includes a tapered portion 1505  
5 having an outer surface 1510, one or more circumferential grooves 1515a and 1515b, and one or more axial grooves 1520a and 1520b.

The circumferential grooves 1515 are fluidically coupled to the axial grooves 1520. In this manner, during the radial expansion process, lubricating fluids are transmitted from the areas in front of the front 1500a and/or behind the rear 1500b  
10 of the expansion cone 1500 into the circumferential grooves 1515. Thus, the trailing edge portion of the interface between the expansion cone 1500 and the tubular member 370 is provided with an increased supply of lubricant, thereby reducing the amount of force required to radially expand the tubular member 370. Furthermore, pressurized lubricating fluids pass from the fluid passages 1520 to the area in front  
15 of the front 1500a of the expansion cone 1500. In this manner, the area adjacent to the front 1500a of the expansion cone 1500 is cleaned of foreign materials. The lubricating fluids are injected into the internal flow passages 1520 by pressurizing the area behind the rear 1500b expansion cone 1500 during the radial expansion process.

20 The expansion cone 1500 includes a plurality of circumferential grooves 1515. The cross sectional area of the circumferential grooves 1515 range from about  $2 \times 10^{-4} \text{ in}^2$  to  $5 \times 10^{-2} \text{ in}^2$  in order to optimally provide lubrication to the trailing edge portion of the interface between the expansion cone 1500 and the tubular member 370 during the radial expansion process. The expansion cone 1500  
25 includes circumferential grooves 1515 that are concentrated about the axial midpoint of the tapered portion 1505 in order to optimally provide lubrication to the trailing edge portion of the interface between the expansion cone 1500 and the tubular member 370 during the radial expansion process. The circumferential grooves 1515 are equally spaced along the trailing edge portion of the expansion cone 1500 in  
30 order to optimally provide lubrication to the trailing edge portion of the interface

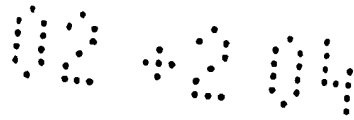
between the expansion cone 1500 and the tubular member 370 during the radial expansion process.

The expansion cone 1500 includes a plurality of axial grooves 1520 coupled to each of the circumferential grooves 1515. The axial grooves 1520 fluidically couple the front end and the rear end of the expansion cone 1500. The cross sectional area of the axial grooves 1520 range from about  $2 \times 10^{-4}$  in<sup>2</sup> to  $5 \times 10^{-2}$  in<sup>2</sup>, respectively, in order to optimally provide lubrication to the trailing edge portion of the interface between the expansion cone 1500 and the tubular member 370 during the radial expansion process. The cross sectional area of the circumferential grooves 1515 is greater than the cross sectional area of the axial grooves 1520 in order to minimize resistance to fluid flow. The axial grooves 1520 are spaced apart in the circumferential direction by at least about 3 inches in order to optimally provide lubrication during the radial expansion process.

Referring to FIG. 16, an expansion cone 1600 is used in the repair apparatus 300 that includes a front end 1600a and a rear end 1600b, includes a tapered portion 1605 having an outer surface 1610, one or more circumferential grooves 1615a and 1615b, and one or more axial grooves 1620a and 1620b.

The circumferential grooves 1615 are fluidically coupled to the axial grooves 1620. In this manner, during the radial expansion process, lubricating fluids are transmitted from the area ahead of the front 1600a of the expansion cone 1600 into the circumferential grooves 1615. Thus, the trailing edge portion of the interface between the expansion cone 1600 and a tubular member is provided with an increased supply of lubricant, thereby reducing the amount of force required to radially expand the tubular member 370. The lubricating fluids are injected into the axial grooves 1620 using a fluid conduit that is coupled to the tapered end 3205 of the expansion cone 1600.

The expansion cone 1600 includes a plurality of circumferential grooves 1615. The cross sectional area of the circumferential grooves 1615 ranges from about  $2 \times 10^{-4}$  in<sup>2</sup> to  $5 \times 10^{-2}$  in<sup>2</sup> in order to optimally provide lubrication to the trailing edge portion of the interface between the expansion cone 1600 and the tubular



member 370 during the radial expansion process. The expansion cone 1600 includes circumferential grooves 1615 that are concentrated about the axial midpoint of the tapered portion 1605 in order to optimally provide lubrication to the trailing edge portion of the interface between the expansion cone 1600 and the tubular member 370 during the radial expansion process. The circumferential grooves 1615 are equally spaced along the trailing edge portion of the expansion cone 1600 in order to optimally provide lubrication to the trailing edge portion of the interface between the expansion cone 1600 and the tubular member 370 during the radial expansion process.

The expansion cone 1600 includes a plurality of axial grooves 1620 coupled to each of the circumferential grooves 1615. The axial grooves 1620 intersect each of the circumferential grooves 1615 at an acute angle. The cross sectional area of the axial grooves 1620 ranges from about  $2 \times 10^{-4} \text{ in}^2$  to  $5 \times 10^{-2} \text{ in}^2$  in order to optimally provide lubrication to the trailing edge portion of the interface between the expansion cone 1600 and the tubular member 370 during the radial expansion process. The cross sectional area of the circumferential grooves 1615 is greater than the cross sectional area of the axial grooves 1620. The axial grooves 1620 are spaced apart in the circumferential direction by at least about 3 inches in order to optimally provide lubrication during the radial expansion process. The axial grooves 1620 intersect the longitudinal axis of the expansion cone 1600 at a larger angle than the angle of attack of the tapered portion 1605 in order to optimally provide lubrication during the radial expansion process.

Referring to FIG. 17, an expansion cone 1700 is used in the repair apparatus 300 that includes a front end 1700a and a rear end 1700b, includes a tapered portion 1705 having an outer surface 1710, a spiral circumferential groove 1715, and one or more internal flow passages 1720.

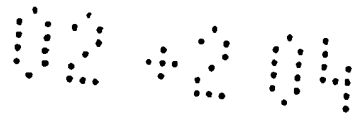
The circumferential groove 1715 is fluidically coupled to the internal flow passage 1720. In this manner, during the radial expansion process, lubricating fluids are transmitted from the area ahead of the front 1700a of the expansion cone 1700 into the circumferential groove 1715. Thus, the trailing edge portion of the interface

between the expansion cone 1700 and the tubular member 370 is provided with an increased supply of lubricant, thereby reducing the amount of force required to radially expand the tubular member. The lubricating fluids are injected into the internal flow passage 1720 using a fluid conduit that is coupled to the tapered end  
 5 1705 of the expansion cone 1700.

The expansion cone 1700 includes a plurality of spiral circumferential grooves 1715. The cross sectional area of the circumferential groove 1715 ranges from about  $2 \times 10^{-4} \text{ in}^2$  to  $5 \times 10^{-2} \text{ in}^2$  in order to optimally provide lubrication to the trailing edge portion of the interface between the expansion cone 1700 and the  
 10 tubular member 370 during the radial expansion process. The expansion cone 1700 includes circumferential grooves 1715 that are concentrated about the axial midpoint of the tapered portion 1705 in order to optimally provide lubrication to the trailing edge portion of the interface between the expansion cone 1700 and the tubular member 370 during the radial expansion process. The circumferential grooves 1715  
 15 are equally spaced along the trailing edge portion of the expansion cone 1700 in order to optimally provide lubrication to the trailing edge portion of the interface between the expansion cone 1700 and the tubular member 370 during the radial expansion process.

The expansion cone 1700 includes a plurality of flow passages 1720 coupled  
 20 to each of the circumferential grooves 1715. The cross-sectional area of the flow passages 1720 ranges from about  $2 \times 10^{-4} \text{ in}^2$  to  $5 \times 10^{-2} \text{ in}^2$  in order to optimally provide lubrication to the trailing edge portion of the interface between the expansion cone 1700 and the tubular member 370 during the radial expansion process. The cross sectional area of the circumferential groove 1715 is greater than  
 25 the cross sectional area of the flow passage 1720 in order to minimize resistance to fluid flow.

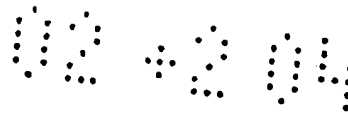
Referring to FIG. 18, an expansion cone 1800 is used in the repair apparatus 300 that includes a front end 1800a and a rear end 1800b, includes a tapered portion 1805 having an outer surface 1810, a spiral circumferential groove 1815, and one or  
 30 more axial grooves 1820a, 1820b and 1820c.



The circumferential groove 1815 is fluidically coupled to the axial grooves 1820. In this manner, during the radial expansion process, lubricating fluids are transmitted from the area ahead of the front 1800a of the expansion cone 1800 into the circumferential groove 1815. Thus, the trailing edge portion of the interface  
5 between the expansion cone 1800 and a tubular member is provided with an increased supply of lubricant, thereby reducing the amount of force required to radially expand the tubular member 370. The lubricating fluids are injected into the axial grooves 1820 using a fluid conduit that is coupled to the tapered end 1805 of the expansion cone 1800.

10 The expansion cone 1800 includes a plurality of spiral circumferential grooves 1815. The cross sectional area of the circumferential grooves 1815 range from about  $2 \times 10^{-4} \text{ in}^2$  to  $5 \times 10^{-2} \text{ in}^2$  in order to optimally provide lubrication to the trailing edge portion of the interface between the expansion cone 1800 and the tubular member 370 during the radial expansion process. The expansion cone 1800  
15 includes circumferential grooves 1815 concentrated about the axial midpoint of the tapered portion 1805 in order to optimally provide lubrication to the trailing edge portion of the interface between the expansion cone 1800 and the tubular member 370 during the radial expansion process. The circumferential grooves 1815 are equally spaced along the trailing edge portion of the expansion cone 1800 in order to  
20 optimally provide lubrication to the trailing edge portion of the interface between the expansion cone 1800 and the tubular member 370 during the radial expansion process.

The expansion cone 1800 includes a plurality of axial grooves 1820 coupled to each of the circumferential grooves 1815. The cross sectional area of the axial  
25 grooves 1820 range from about  $2 \times 10^{-4} \text{ in}^2$  to  $5 \times 10^{-2} \text{ in}^2$  in order to optimally provide lubrication to the trailing edge portion of the interface between the expansion cone 1800 and the tubular member 370 during the radial expansion process. The axial grooves 1820 intersect the circumferential grooves 1815 in a perpendicular manner. The cross sectional area of the circumferential groove 1815  
30 is greater than the cross sectional area of the axial grooves 1820 in order to



minimize resistance to fluid flow. The circumferential spacing of the axial grooves is greater than about 3 inches in order to optimally provide lubrication during the radial expansion process. The axial grooves 1820 intersect the longitudinal axis of the expansion cone at an angle greater than the angle of attack of the tapered portion 1805 in order to optimally provide lubrication during the radial expansion process.

Referring to FIG. 19, an expansion cone 1900 is used in the repair apparatus 300 that includes a front end 1900a and a rear end 1900b, includes a tapered portion 1905 having an outer surface 1910, a circumferential groove 1915, a first axial groove 1920, and one or more second axial grooves 1925a, 1925b, 1925c and 1925d.

The circumferential groove 1915 is fluidically coupled to the axial grooves 1920 and 1925. In this manner, during the radial expansion process, lubricating fluids are preferably transmitted from the area behind the back 1900b of the expansion cone 1900 into the circumferential groove 1915. Thus, the trailing edge portion of the interface between the expansion cone 1900 and the tubular member 370 is provided with an increased supply of lubricant, thereby reducing the amount of force required to radially expand the tubular member 370. The lubricating fluids are injected into the first axial groove 1920 by pressurizing the region behind the back 1900b of the expansion cone 1900. The lubricant is further transmitted into the second axial grooves 1925 where the lubricant preferably cleans foreign materials from the tapered portion 1905 of the expansion cone 1900.

The expansion cone 1900 includes a plurality of circumferential grooves 1915. The cross sectional area of the circumferential groove 1915 ranges from about  $2 \times 10^{-4} \text{ in}^2$  to  $5 \times 10^{-2} \text{ in}^2$  in order to optimally provide lubrication to the trailing edge portion of the interface between the expansion cone 1900 and the tubular member 370 during the radial expansion process. The expansion cone 1900 includes circumferential grooves 1915 concentrated about the axial midpoint of the tapered portion 1905 in order to optimally provide lubrication to the trailing edge portion of the interface between the expansion cone 1900 and the tubular member 370 during the radial expansion process. The circumferential grooves 1915 are



equally spaced along the trailing edge portion of the expansion cone 1900 in order to optimally provide lubrication to the trailing edge portion of the interface between the expansion cone 1900 and the tubular member 370 during the radial expansion process.

5           The expansion cone 1900 includes a plurality of first axial grooves 1920 coupled to each of the circumferential grooves 1915. The first axial grooves 1920 extend from the back 1900b of the expansion cone 1900 and intersect the circumferential groove 1915. The cross sectional area of the first axial groove 1920 ranges from about  $2 \times 10^{-4} \text{ in}^2$  to  $5 \times 10^{-2} \text{ in}^2$  in order to optimally provide lubrication  
10 to the trailing edge portion of the interface between the expansion cone 1900 and the tubular member 370 during the radial expansion process. The first axial groove 1920 intersects the circumferential groove 1915 in a perpendicular manner. The cross sectional area of the circumferential groove 1915 is greater than the cross sectional area of the first axial groove 1920 in order to minimize resistance to fluid  
15 flow. The circumferential spacing of the first axial grooves 1920 is greater than about 3 inches in order to optimally provide lubrication during the radial expansion process.

          The expansion cone 1900 includes a plurality of second axial grooves 1925 coupled to each of the circumferential grooves 1915. The second axial grooves  
20 1925 extend from the front 1900a of the expansion cone 1900 and intersect the circumferential groove 1915. The cross sectional area of the second axial grooves 1925 ranges from about  $2 \times 10^{-4} \text{ in}^2$  to  $5 \times 10^{-2} \text{ in}^2$  in order to optimally provide lubrication to the trailing edge portion of the interface between the expansion cone 1900 and the tubular member 370 during the radial expansion process. The second  
25 axial grooves 1925 intersect the circumferential groove 1915 in a perpendicular manner. The cross sectional area of the circumferential groove 1915 is greater than the cross sectional area of the second axial grooves 1925 in order to minimize resistance to fluid flow. The circumferential spacing of the second axial grooves 1925 is greater than about 3 inches in order to optimally provide lubrication during  
30 the radial expansion process. The second axial grooves 1925 intersect the

longitudinal axis of the expansion cone 1900 at an angle greater than the angle of attack of the tapered portion 1905 in order to optimally provide lubrication during the radial expansion process.

Referring to Fig. 20, The first axial groove 1920 includes a first portion 2005  
5 having a first radius of curvature 2010, a second portion 2015 having a second radius of curvature 2020, and a third portion 2025 having a third radius of curvature 2030. The radius of curvatures, 2010, 2020 and 2030 are substantially equal. The radius of curvatures, 2010, 2020 and 2030 are all substantially equal to 0.0625 inches.

10 Referring to Fig. 21, The circumferential groove 1915 includes a first portion 2105 having a first radius of curvature 2110, a second portion 2115 having a second radius of curvature 2120, and a third portion 2125 having a third radius of curvature 2130. The radius of curvatures, 2110, 2120 and 2130 are substantially equal. The radius of curvatures, 2110, 2120 and 2130 are all substantially equal to 0.125 inches.

15 Referring to Fig. 22, The second axial groove 1925 includes a first portion 2205 having a first radius of curvature 2210, a second portion 2215 having a second radius of curvature 2220, and a third portion 2225 having a third radius of curvature 2230. The first radius of curvature 2210 is greater than the third radius of curvature 2230. The first radius of curvature 2210 is equal to 0.5 inches, the second radius of  
20 curvature 2220 is equal to 0.0625 inches, and the third radius of curvature 2230 is equal to 0.125 inches.

Referring to Fig. 23, an expansion cone 2300 is used in the repair apparatus 300 that includes an internal flow passage 2305 having an insert 2310 including a flow passage 2315. The cross sectional area of the flow passage 2315 is less than  
25 the cross sectional area of the flow passage 2305. More generally, A plurality of inserts 2310 are provided, each with different sizes of flow passages 2315. In this manner, the flow passage 2305 is machined to a standard size, and the lubricant supply is varied by using different sized inserts 2310. The teachings of the expansion cone 2300 are incorporated into the expansion cones 1200, 1300, 1400,  
30 and 1700.

Referring to Fig. 24, The insert 2310 includes a filter 2405 for filtering particles and other foreign materials from the lubricant that passes into the flow passage 2305. In this manner, the foreign materials are prevented from clogging the flow passage 2305 and other flow passages within the expansion cone 2300.

5        The increased lubrication provided to the trailing edge portion of the expansion cones 1200, 1300, 1400, 1500, 1600, 1700, 1800, and 1900 greatly reduces the amount of galling or seizure caused by the interface between the expansion cones and the tubular member 370 during the radial expansion process thereby permitting larger continuous sections of tubulars to be radially expanded in a  
10 single continuous operation. Thus, use of the expansion cones 1200, 1300, 1400, 1500, 1600, 1700, 1800, and 1900 reduces the operating pressures required for radial expansion and thereby reduces the size of the pump 325. In addition, failure, bursting, and/or buckling of the tubular member 370 during the radial expansion process is significantly reduced, and the success ratio of the radial expansion process  
15 is greatly increased.

      The lubricating fluids used with the expansion cones 1200, 1300, 1400, 1500, 1600, 1700, 1800 and 1900 for expanding the tubular member 370 have viscosities ranging from about 1 to 10,000 centipoise in order to optimize the injection of the lubricating fluids into the circumferential grooves of the expansion cones during the  
20 radial expansion process. The lubricating fluids used with the expansion cones 1200, 1300, 1400, 1500, 1600, 1700, 1800 and 1900 for expanding the tubular member 370 comprise various conventional lubricants available from various commercial vendors consistent with the teachings of the present disclosure in order to optimize the injection of the lubricating fluids into the circumferential grooves of  
25 the expansion cones during the radial expansion process.

      As illustrated in FIG. 25, the expansion cone 375 further includes a central passage 2505 for receiving the support member 340 and the repair apparatus 300 further includes one or more sealing members 2510 and one or more bearing members 2515.

The sealing members 2510 are preferably adapted to fluidically seal the dynamic interface between the central passage 2505 of the expansion cone 375 and the support member 340. The sealing members 2510 may be any number of conventional commercially available sealing members. The sealing members 2510  
 5 are conventional O-rings sealing members available from various commercial suppliers in order to optimally provide a fluidic seal.

The bearing members 2515 are preferably adapted to provide a sliding interface between the central passage 2505 of the expansion cone 375 and the support member 340. The bearing members 2515 may be any number of  
 10 conventional commercially available bearings. The bearing members 2515 are wear bands available from Haliburton Energy Services in order to optimally provide a sliding interface that minimizes wear.

The sealing member 380 is coupled to the exterior surface of the expandable tubular member 375. The sealing member 380 is preferably adapted to fluidically seal  
 15 the interface between the expandable tubular member 375 and the wellbore casing 100 after the radial expansion of the expandable tubular member 375. The sealing member 380 may be any number of conventional commercially available sealing members. The sealing member 380 is a nitrile rubber sealing member available from Eustler, Inc. in order to optimally provide a high pressure, high load bearing  
 20 seal between the expandable tubular member 375 and the casing 100.

As illustrated in FIG. 3a, During placement of the repair apparatus 300 within the wellbore casing 100, the repair apparatus 300 is supported by the support member 305. During placement of the repair apparatus 300 within the wellbore casing 100, fluidic materials within the wellbore casing 100 are conveyed to a  
 25 location above the repair apparatus 300 using the fluid conduits 335, 345, and 355. In this manner, surge pressures during placement of the repair apparatus 300 within the wellbore casing 100 are minimized.

Prior to placement of the repair apparatus 300 in the wellbore, the outer surfaces of the repair apparatus 300 are coated with a lubricating fluid to facilitate  
 30 their placement the wellbore and reduce surge pressures. The lubricating fluid

comprises BARO-LUB GOLD-SEAL<sup>(RTM)</sup> brand drilling mud lubricant, available from Baroid<sup>(RTM)</sup> Drilling Fluids, Inc. In this manner, the insertion of the repair apparatus 300 into the wellbore casing 100 is optimized.

After placement of the repair apparatus 300 within the wellbore casing 100, in step 210, the logging tool 310 is used in a conventional manner to locate the openings 115 in the wellbore casing 100.

Once the openings 115 have been located by the logging tool 310, in step 215, the repair apparatus 300 is further positioned within the wellbore casing 100 with the sealing member 380 placed in opposition to the openings 115.

As illustrated in FIGS. 3b and 3c, After the repair apparatus 300 has been positioned with the sealing member 380 in opposition to the openings 115, in step 220, the tubular member 370 is radially expanded into contact with the wellbore casing 100. The tubular member 370 is radially expanded by displacing the expansion cone 375 in the axial direction. The expansion cone 375 is displaced in the axial direction by pressurizing the interior portion 385. The interior portion 385 is pressurized by pumping fluidic materials into the interior portion 385 using the pump 325.

The pump 325 pumps fluidic materials from the region above and proximate to the repair apparatus 300 into the interior portion 385 using the fluidic passages 320 and 330. In this manner, the interior portion 385 is pressurized and the expansion cone 375 is displaced in the axial direction. In this manner, the tubular member 370 is radially expanded into contact with the wellbore casing 100. The interior portion 385 is pressurized to operating pressures ranging from about 0 to 12,000 psi using flow rates ranging from about 0 to 500 gallons/minute. Fluidic materials displaced by the axial movement of the expansion cone 375 are conveyed to a location above the repair apparatus 300 by the fluid conduits 335, 345, and 355. During the pumping of fluidic materials into the interior portion 385 by the pump 325, the tubular member 370 is maintained in a substantially stationary position.

As illustrated in FIG. 3d, after the completion of the radial expansion of the tubular member 370, the locking member 365 is decoupled from the tubular member

370 and the repair apparatus 300 is removed from the wellbore casing 100. During the removal of the repair apparatus 300 from the wellbore casing 100, fluidic materials above the repair apparatus 300 are conveyed to a location below the repair apparatus 300 using the fluid conduits 335, 345 and 355. In this manner, the removal of the repair apparatus 300 from the wellbore casing is facilitated.

As illustrated in FIG. 3e, The openings 115 in the wellbore casing 100 are sealed off by the radially expanded tubular member 370 and the sealing member 380. In this manner, the repair apparatus 300 provides a compact and efficient device for repairing wellbore casings. More generally, the repair apparatus 300 is used to repair and form wellbore casings, pipelines, and structural supports.

Referring to FIG. 26a, in step 205, a repair apparatus 2600 is positioned within the wellbore casing 100.

The repair apparatus 2600 preferably includes a first support member 2605, a logging tool 2610, a housing 2615, a first fluid conduit 2620, a pump 2625, a second fluid conduit 2630, a first valve 2635, a third fluid conduit 2640, a second valve 2645, a fourth fluid conduit 2650, a second support member 2655, a fifth fluid conduit 2660, the third support member 2665, a sixth fluid conduit 2670, sealing members 2675, a locking member 2680, an expandable tubular 2685, an expansion cone 2690, a sealing member 2695, a packer 2700, a seventh fluid conduit 2705, and a third valve 2710.

The first support member 2605 is preferably coupled to the logging tool 2610 and the housing 2615. The first support member 2605 is preferably adapted to be coupled to and supported by a conventional support member such as, for example, a wireline or a drill string. The first support member 2605 preferably has a substantially annular cross section in order to provide one or more conduits for conveying fluidic materials from the apparatus 2600. The first support member 2605 is further preferably adapted to convey electrical power and communication signals to the logging tool 2610, the pump 2625, the valves 2635, 2645, and 2710, and the packer 2700.

The logging tool 2610 is preferably coupled to the first support member 2605. The logging tool 2610 is preferably adapted to detect defects in the wellbore casing 100. The logging tool 2610 may be any number of conventional commercially available logging tools suitable for detecting defects in wellbore casings, pipelines, or structural supports. The logging tool 2610 is a CAST logging tool, available from Halliburton<sup>(RTM)</sup> Energy Services in order to optimally provide detection of defects in the wellbore casing 100. The logging tool 2610 is contained within the housing 2615 in order to provide a repair apparatus 2600 that is rugged and compact.

10 The housing 2615 is preferably coupled to the first support member 2605, the second support member 2655, the sealing members 2675, and the locking member 2680. The housing 2615 is preferably releasably coupled to the tubular member 2685. The housing 2615 is further preferably adapted to contain and support the logging tool 2610 and the pump 2625.

15 The first fluid conduit 2620 is preferably fluidically coupled to the inlet of the pump 2625, the exterior region above the housing 2615, and the second fluid conduit 2630. The first fluid conduit 2620 may be contained within the first support member 2605 and the housing 2615. The first fluid conduit 2620 is preferably adapted to convey fluidic materials such as, for example, drilling muds, water, and lubricants at operating pressures and flow rates ranging from about 0 to 12,000 psi and 0 to 500 gallons/minute in order to optimally propagate the expansion cone 2690.

20 The pump 2625 is fluidically coupled to the first fluid conduit 2620 and the third fluid conduit 2640. The pump 2625 is further preferably contained within and support by the housing 2615. The pump 2625 is preferably adapted to convey fluidic materials from the first fluid conduit 2620 to the third fluid conduit 2640 at operating pressures and flow rates ranging from about 0 to 12,000 psi and 0 to 500 gallons/minute in order to optimally provide operating pressure for propagating the expansion cone 2690. The pump 2625 may be any number of conventional commercially available pumps. The pump 2625 is a flow control pump out section,

available from Halliburton<sup>(RTM)</sup> Energy Services in order to optimally provide fluid pressure for propagating the expansion cone 2690. The pump 2625 is preferably adapted to pressurize an interior portion 2715 of the expandable tubular member 2685 to operating pressures ranging from about 0 to 12,000 psi.

5        The second fluid conduit 2630 is fluidically coupled to the first fluid conduit 2620 and the third fluid conduit 2640. The second fluid conduit 2630 is further preferably contained within the housing 2615. The second fluid conduit 2630 is preferably adapted to convey fluidic materials such as, for example, drilling muds, water, and lubricants at operating pressures and flow rates ranging from about 0 to  
10    12,000 psi and 0 to 500 gallons/minute in order to optimally provide propagation of the expansion cone 2690.

      The first valve 2635 is preferably adapted to controllably block the second fluid conduit 2630. In this manner, the flow of fluidic materials through the second fluid conduit 2630 is controlled. The first valve 2635 may be any number of  
15    conventional commercially available flow control valves. The first valve 2635 is a conventional ball valve available from various commercial suppliers.

      The third fluid conduit 2640 is fluidically coupled to the outlet of the pump 2625, the second fluid conduit 2630, and the fifth fluid conduit 2660. The third fluid conduit 2640 is further preferably contained within the housing 2615. The third  
20    fluid conduit 2640 is preferably adapted to convey fluidic materials such as, for example, drilling muds, water, and lubricants at operating pressures and flow rates ranging from about 0 to 12,000 psi and 0 to 500 gallons/minute in order to optimally provide propagation of the expansion cone 2690.

      The second valve 2645 is preferably adapted to controllably block the third  
25    fluid conduit 2640. In this manner, the flow of fluidic materials through the third fluid conduit 2640 is controlled. The second valve 2645 may be any number of conventional commercially available flow control valves. The second valve 2645 is a conventional ball valve available from various commercial sources.

      The fourth fluid conduit 2650 is fluidically coupled to the exterior region  
30    above the housing 2615 and the interior region 2720 within the expandable tubular



member 2685. The fourth fluid conduit 2650 is further preferably contained within the housing 2615. The fourth fluid conduit 2650 is preferably adapted to convey fluidic materials such as, for example, drilling muds, water, and lubricants at operating pressures and flow rates ranging from about 0 to 5,000 psi and 0 to 500  
5 gallons/minute in order to optimally vent fluidic materials in front of the expansion cone 2690 during the radial expansion process.

The second support member 2655 is coupled to the housing 2615 and the third support member 2665. The second support member 2655 is further preferably movably and sealingly coupled to the expansion cone 2690. The second support  
10 member 2655 preferably has a substantially annular cross section in order to provide one or more conduits for conveying fluidic materials. The second support member 2655 is centrally positioned within the expandable tubular member 2685.

The fifth fluid conduit 2660 is fluidically coupled to the third fluid conduit 2640 and the sixth fluid conduit 2670. The fifth fluid conduit 2660 is further  
15 preferably contained within the second support member 2655. The fifth fluid conduit 2660 is preferably adapted to convey fluidic materials such as, for example, drilling muds, water, and lubricants at operating pressures and flow rates ranging from about 0 to 12,000 psi and 0 to 500 gallons/minute in order to optimally propagate the expansion cone 2690.

20 The third support member 2665 is coupled to the second support member 2655. The third support member 2665 is further preferably adapted to support the expansion cone 2690. The third support member 2665 preferably has a substantially annular cross section in order to provide one or more conduits for conveying fluidic materials.

25 The sixth fluid conduit 2670 is fluidically coupled to the fifth fluid conduit 2660 and the interior region 2715 of the expandable tubular member 2685 below the expansion cone 2690. The sixth fluid conduit 2670 is further preferably contained within the third support member 2665. The sixth fluid conduit 2670 is preferably adapted to convey fluidic materials such as, for example, drilling muds, water, and  
30 lubricants at operating pressures and flow rates ranging from about 0 to 12,000 psi

and 0 to 500 gallons/minute in order to optimally propagate the expansion cone 2690.

The sealing members 2675 are preferably coupled to the housing 2615. The sealing members 2675 are preferably adapted to seal the interface between the exterior surface of the housing 2615 and the interior surface of the expandable tubular member 2685. In this manner, the interior portion 2730 of the expandable tubular member 2685 is fluidically isolated from the exterior region above the housing 2615. The sealing members 2675 may be any number of conventional commercially available sealing members. The sealing members 2675 are conventional O-ring sealing members available from various commercial suppliers in order to optimally provide a pressure seal.

The locking member 2680 is preferably coupled to the housing 2615. The locking member 2680 is further preferably releasably coupled to the expandable tubular member 2685. In this manner, the housing 2615 is controllably coupled to the expandable tubular member 2685. In this manner, the housing 2615 is preferably released from the expandable tubular member 2685 upon the completion of the radial expansion of the expandable tubular member 2685. The locking member 2680 may be any number of conventional commercially available releasable locking members. The locking member 2680 is a hydraulically released slip available from various commercial vendors in order to optimally provide support during the radial expansion process.

The locking member 2680 is replaced by or supplemented by one or more conventional shear pins in order to provide an alternative means of controllably releasing the housing 2615 from the expandable tubular member 2685.

The seals 2675 and locking member 2680 are omitted.

The expandable tubular member 2685 is releasably coupled to the locking member 2680. The expandable tubular member 2685 is preferably adapted to be radially expanded by the axial displacement of the expansion cone 2690. The expandable tubular member 2685 is substantially identical to the expandable tubular member 370 described above with reference to the repair apparatus 300.



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